

# Hybrid Oxidizer Tank Filling and Dynamics

Team #3

April 25, 2020

Advisor:

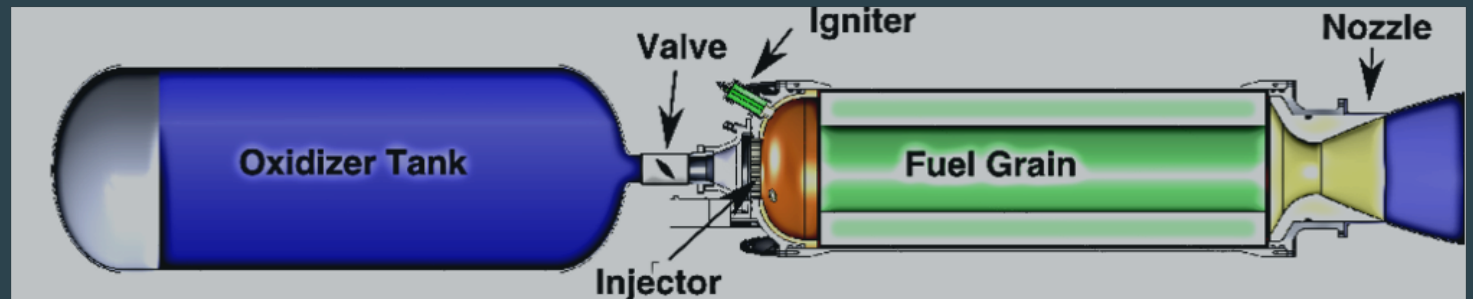
Dr. Jeffrey Santner

# Agenda

- Introduction
- Python Simulations
- ANSYS Simulations
- Structural Design & Analysis
- Control Systems
- Conclusion

# Introduction

- Sub-unit of the Eagle Rocketry SEDS Club at Cal State LA
  - Aerospace centric club researching rocket engines: Hybrid
  - SEDS: Students for Exploration and Development of Space



# Objectives

- Remotely transfer 15kg of nitrous oxide from a commercially purchased vessel
- Create a testing structure for the nitrous oxide pressure vessel which can double as a launch pad
- Predict the time taken for the filling process



# Meet the Team



Orifiel Ortiz

- Computational Fluid Analyst

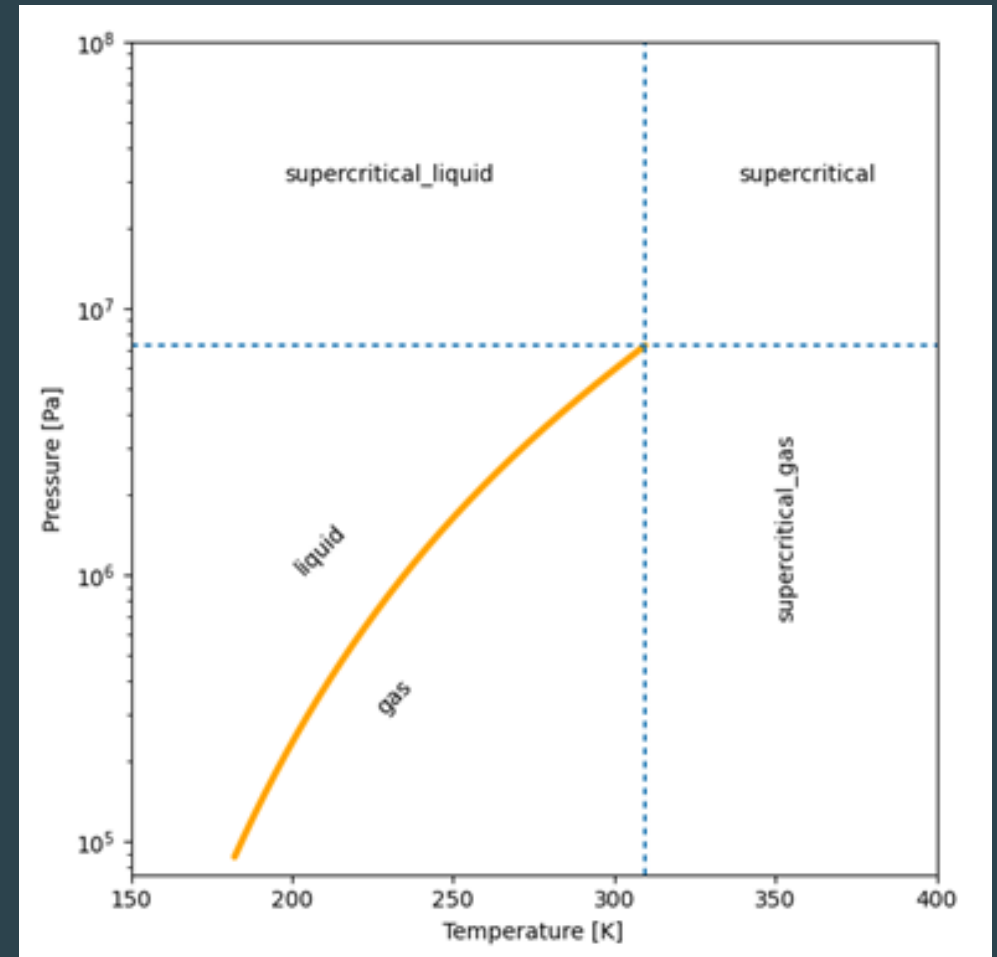
# Python Simulations

# Model Overview

- Based on a modified Equilibrium Model. [1]

Limitations:

- CoolProp cannot compute thermodynamic properties when the fluid is supercritical
- CoolProp cannot calculate  $N_2O$  thermal conductivity or viscosity at any temperature or pressure



# Thermal Conductivity and Viscosity

- Supercritical Thermal Conductivity is calculated from tables published by Richter and Sage. [2]
- Supercritical Viscosity is calculated from work by Takashi et. al. [3]

Validity range:

$277.59K - 444.26K$

$101.33kPa - 34.47 MPa$

Validity range:

$298.15K - 398.15K$

$102.50kPa - 5.50MPa$



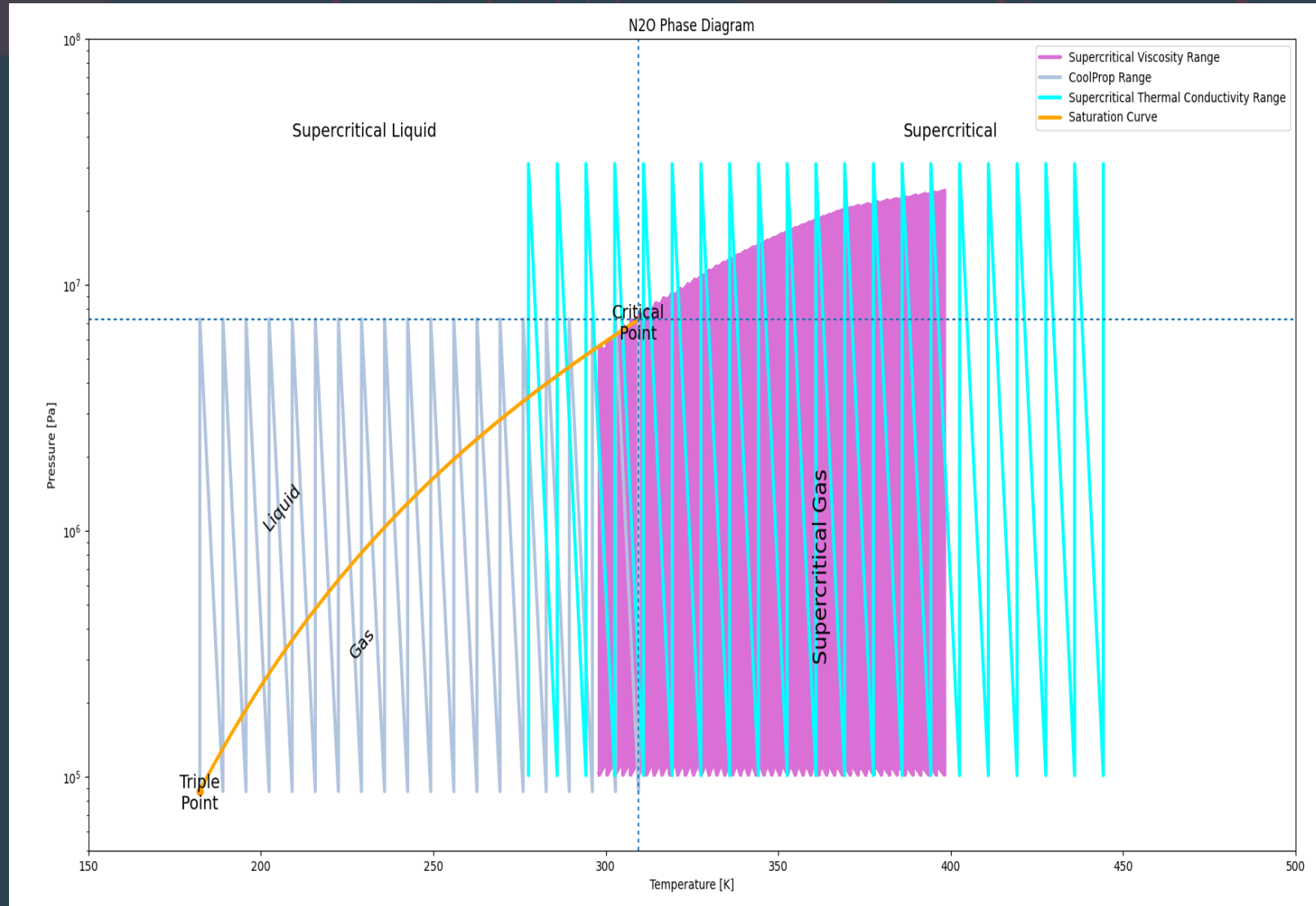
# Thermal Conductivity and Viscosity

- Saturation values calculated from equations published by ESDU. [4]

Validity range:

$182.33K - 309.57K$

$87.3kPa - 7.24MPa$



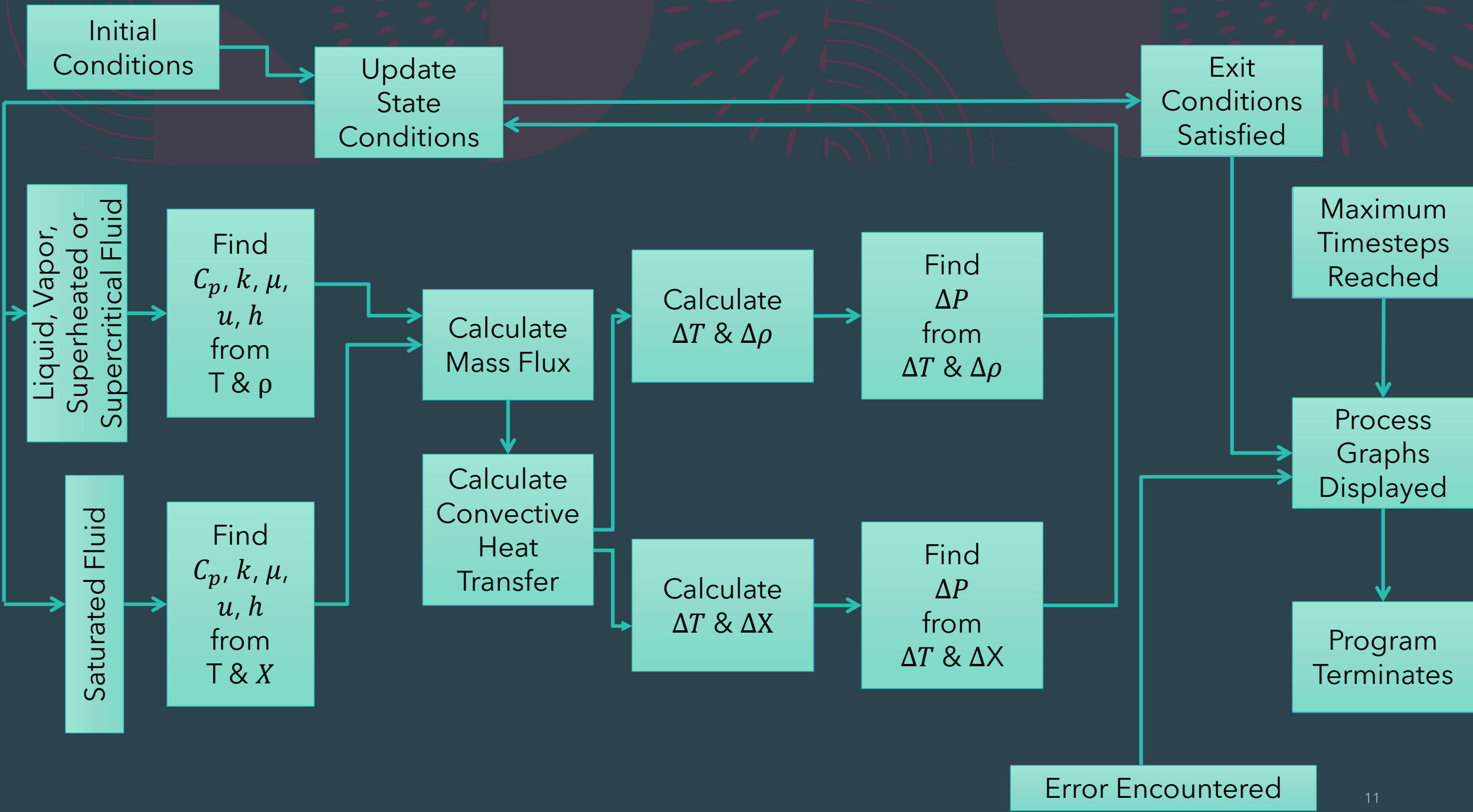
# Solution

- Helmholtz Functional Equations using temperature and density as the two known supercritical fluid states. [5]

$$\alpha(\delta, \tau) = \alpha^0(\delta, \tau) + \alpha^r(\delta, \tau)$$

- $\alpha(\delta, \tau)$  = Helmholtz Energy
- $\delta$  = normalized density
- $\tau$  = normalized temperature

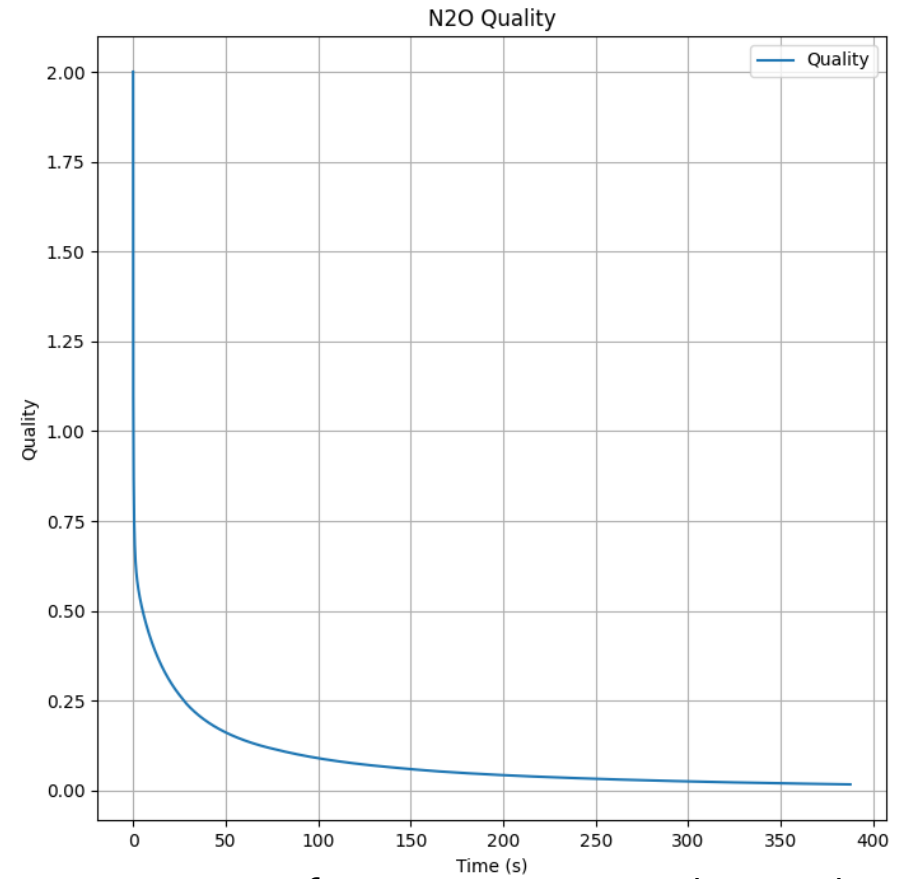
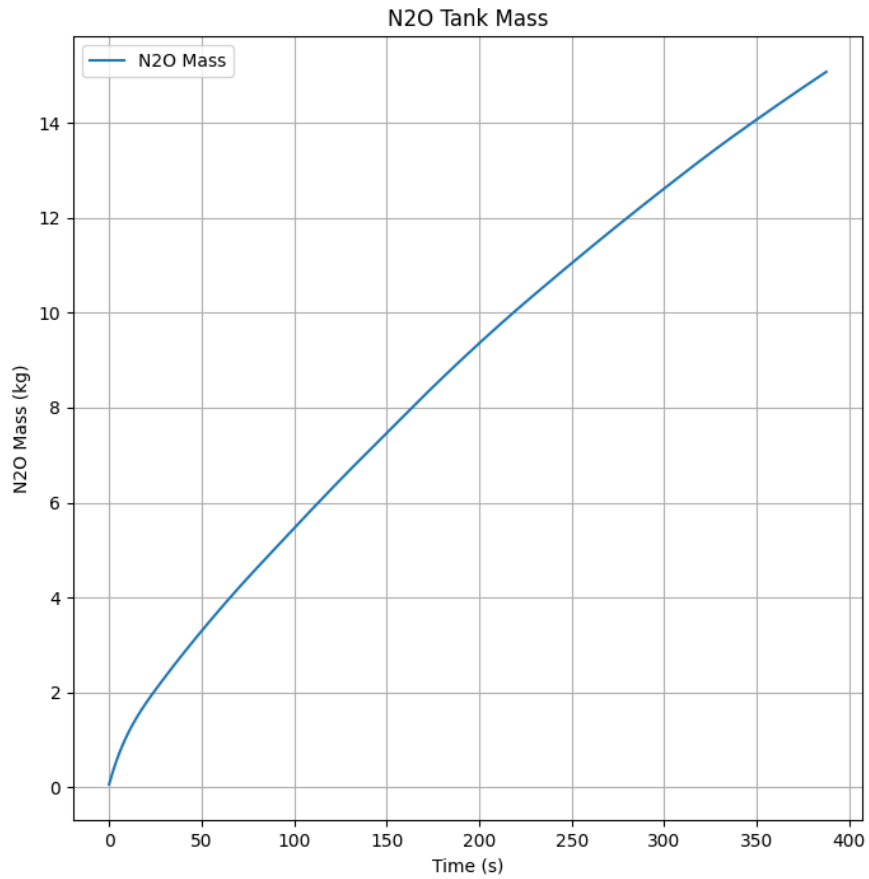
- Other fluid properties can be calculated through the First Law of Thermodynamics, with a known Helmholtz Energy



# Boundary Conditions

- Initial/Atmospheric Temperature:  $298K$
- Initial Pressure: air at  $101.33\text{ kPa}$
- Tank Length:  $1.27\text{ m}$  ( $50\text{ in}$ )
- Tank Inner Diameter:  $152.4\text{ mm}$  ( $6\text{ in}$ )
- Source Tank Pressure:  $689.48\text{ kPa}$  ( $1000\text{ Psi}$ )
- Total Tank Mass:  $15\text{ kg}$

# $m$ & $X$



Note: If  $X > 1$ ,  $N_2O$  superheated

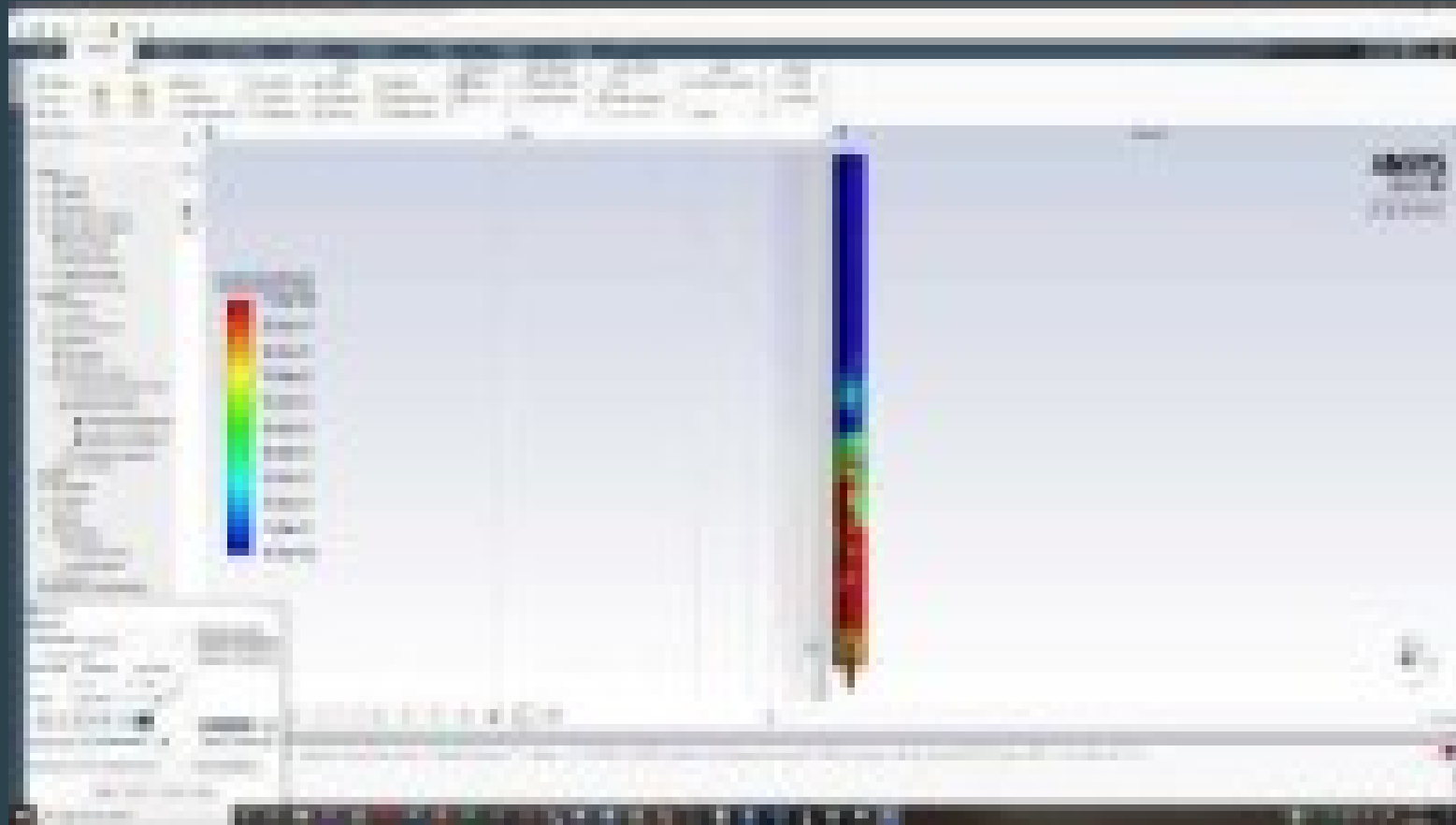


Miguel Romero

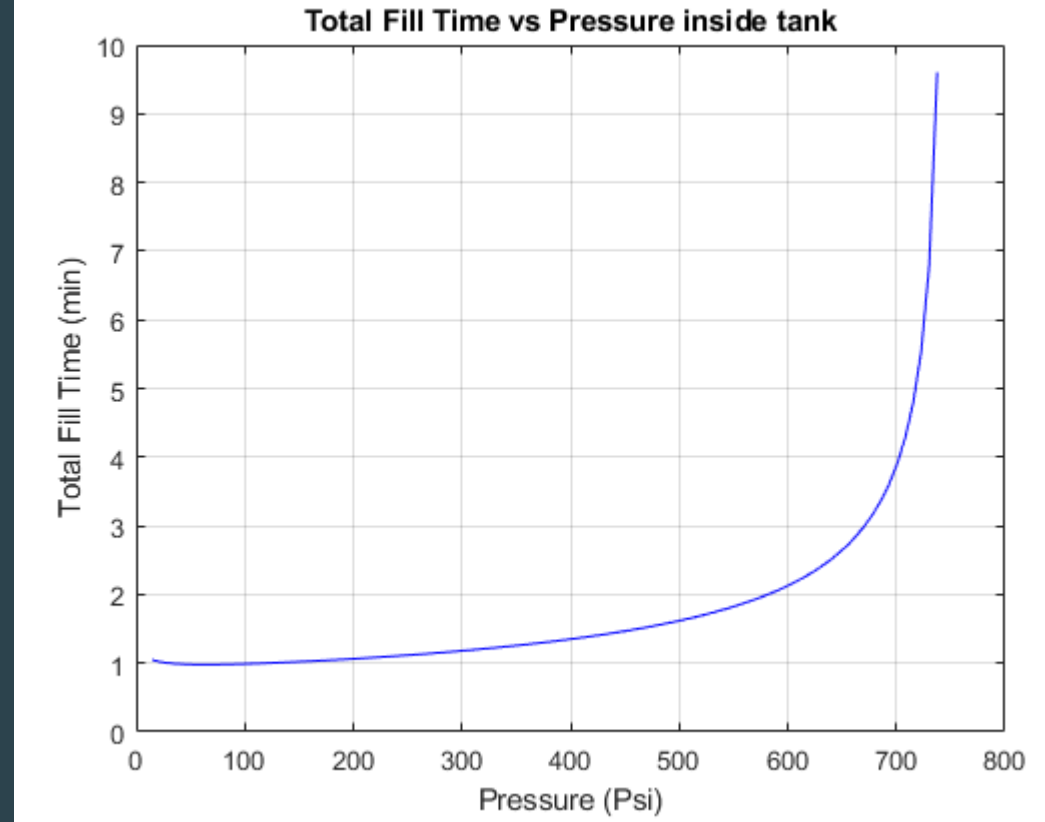
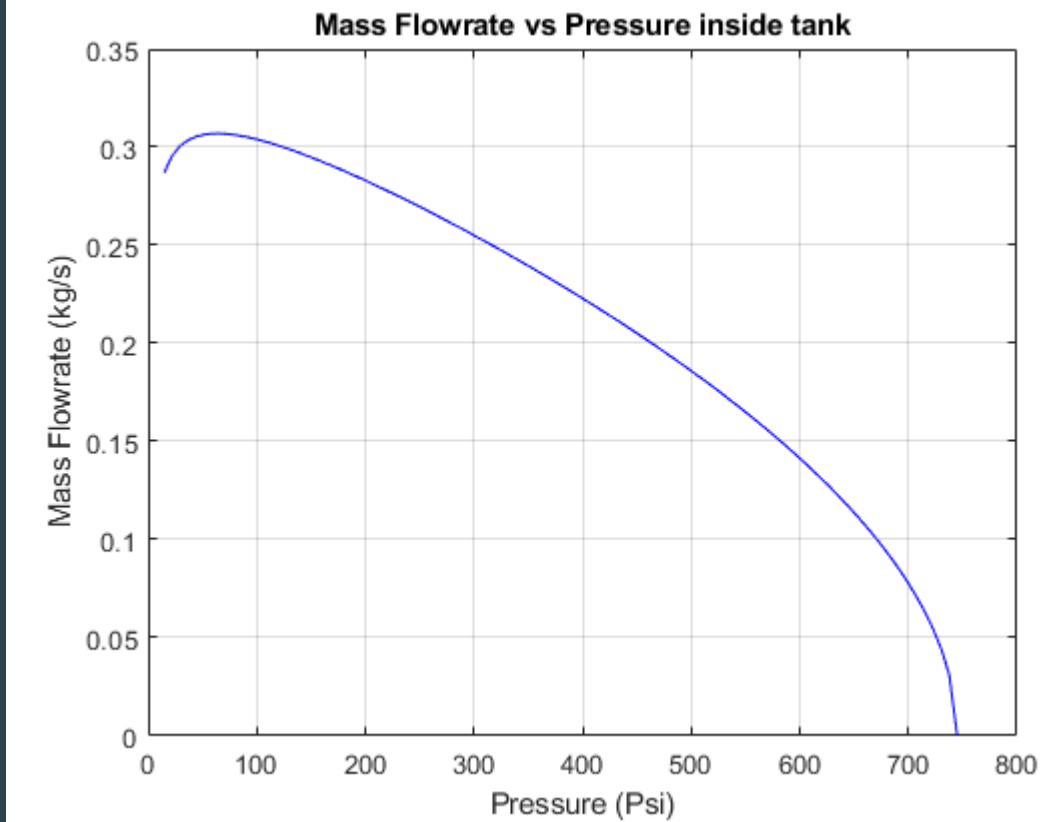
- Pressure Vessel Fluid Analyst

# ANSYS Pressure Vessel Simulations

# Oxidizer Tank Filling



# MATLAB

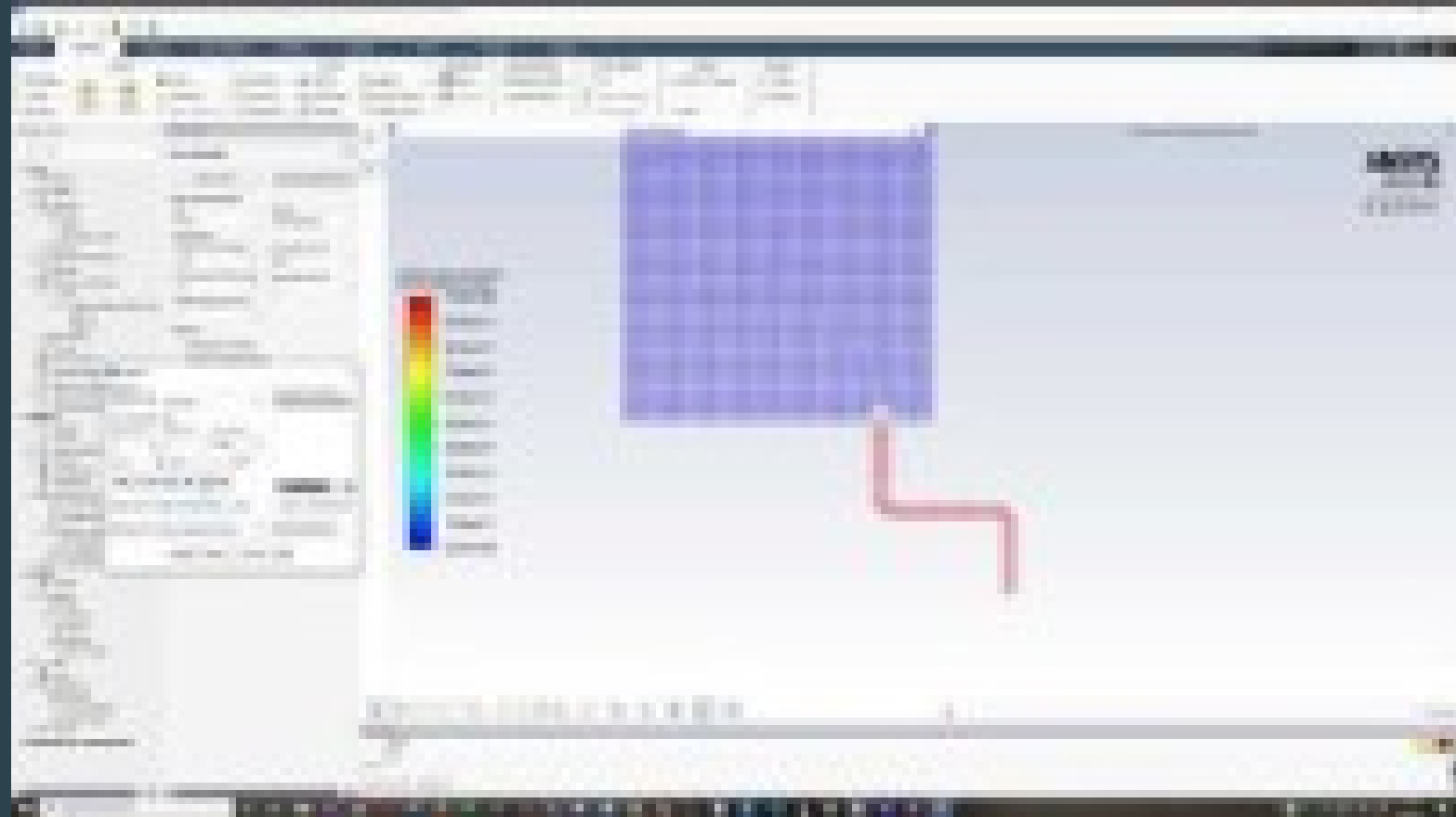
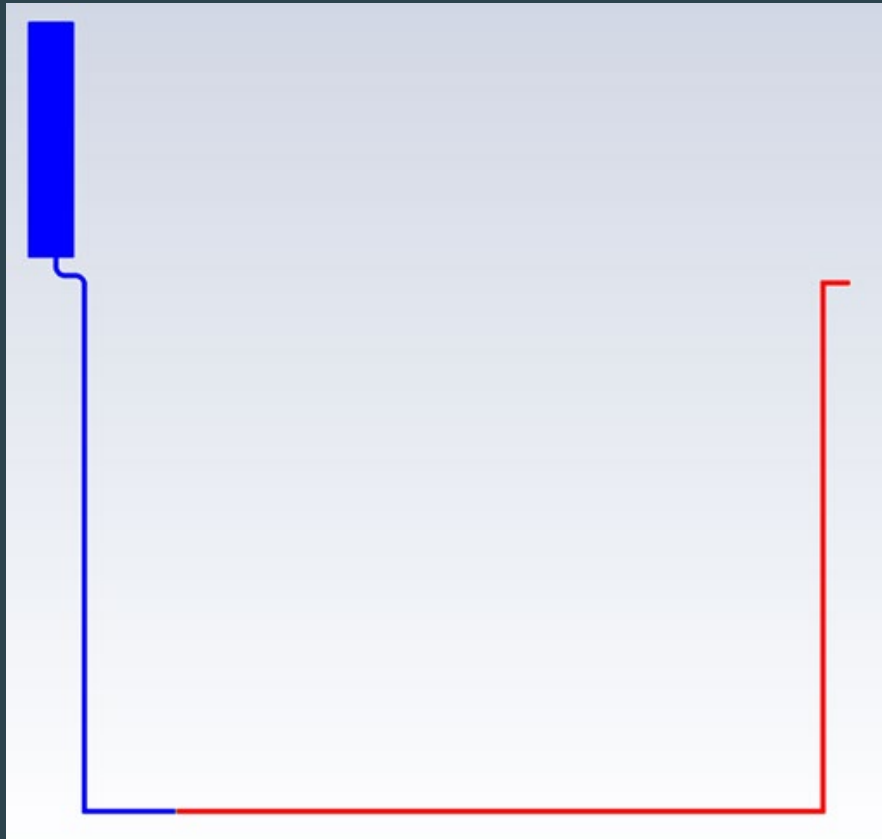




# Ground Fill System

- Inlet diameter to Oxidizer Tank: 0.25in
- Inner diameter of piping system is 0.25in
- Going to open the source tank at varying pressures to slow down the filling time

# Ground Fill System



# Mass Requirements

- Do we have enough N2O to fill the tank and for the ground system piping?
  - N2O mass system = 28kg
  - N2O source tank = 27kg
  - 2 source tanks from vendor

Grade & Gas Specification	Cyl. Size	Product Code	Contents Pounds	Cylinder Rental	Cylinder Pressure (psi)	CGA Outlet	Nominal Cylinder Dim. (in.)	Shipping Wt. Lbs
Nitrous Oxide AA 99.6%	200	NO AA200	60	YES	745	326	9 x 55	202
	G	NO AA56	56	YES	745	326	8.5 x 55	170
	G	NO AA50	50	YES	745	326	8.5 x 55	164

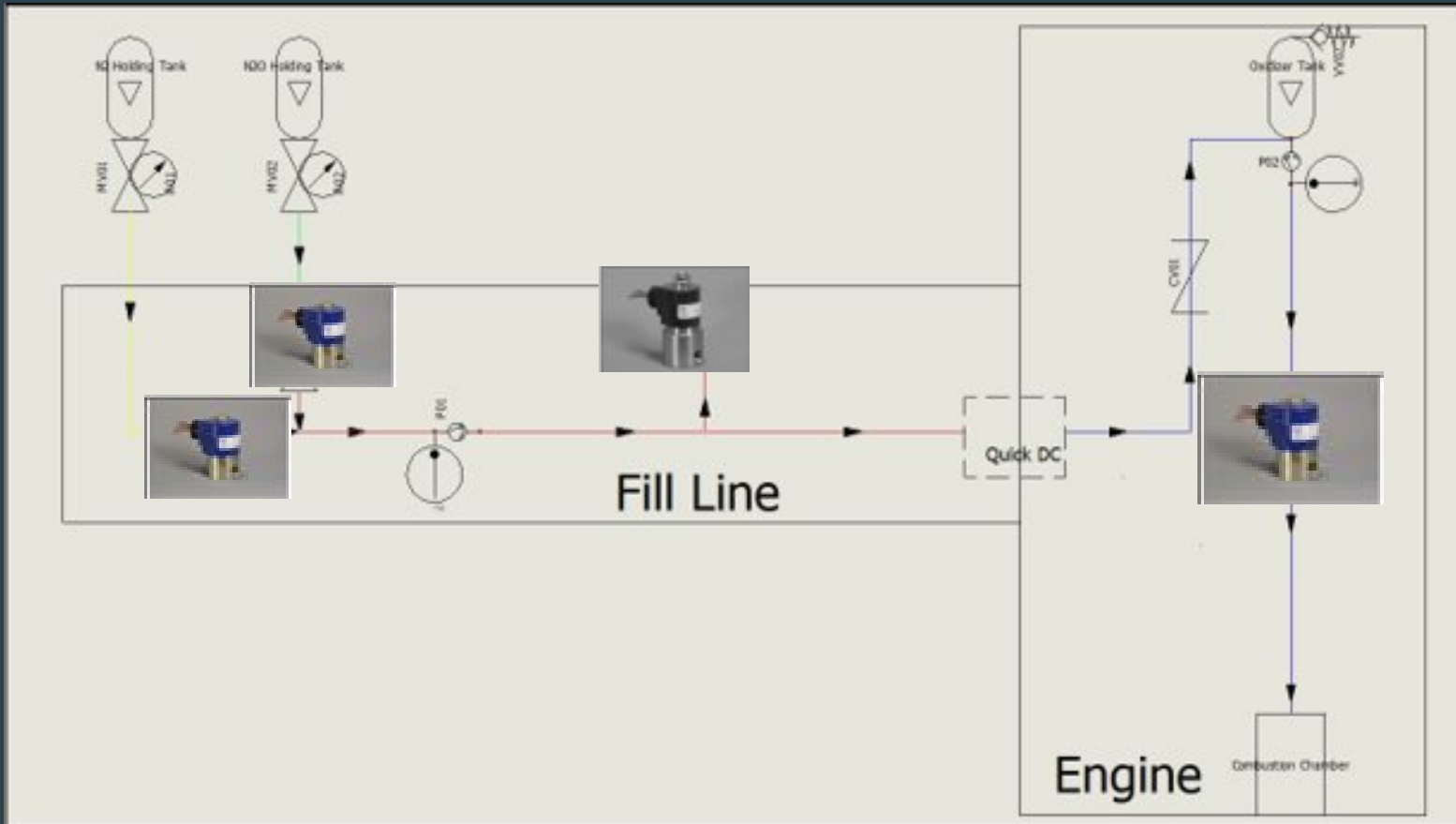
# Anslys & Valve Simulations



Amy Moore

- Piping & Valve Fluid Analyst

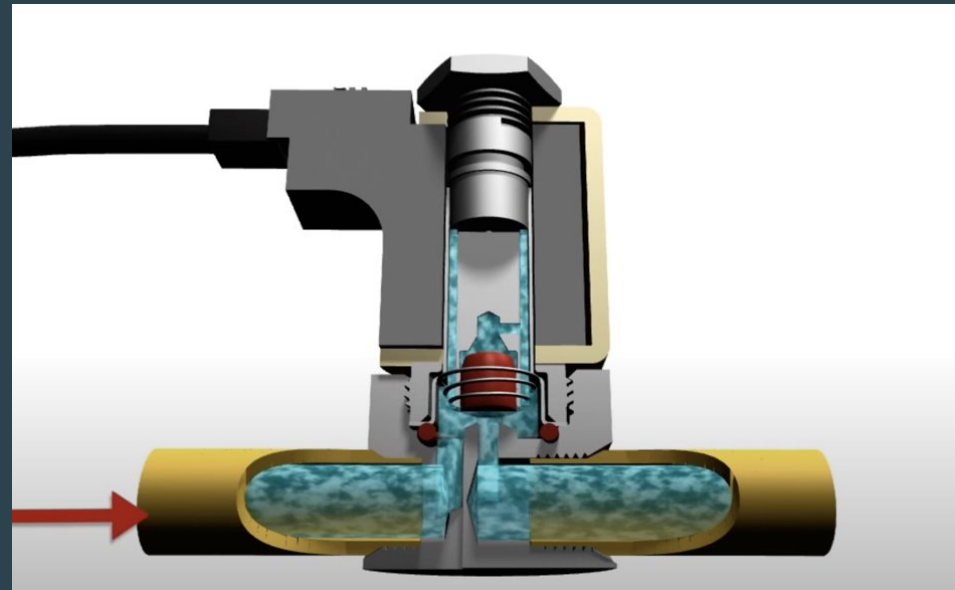
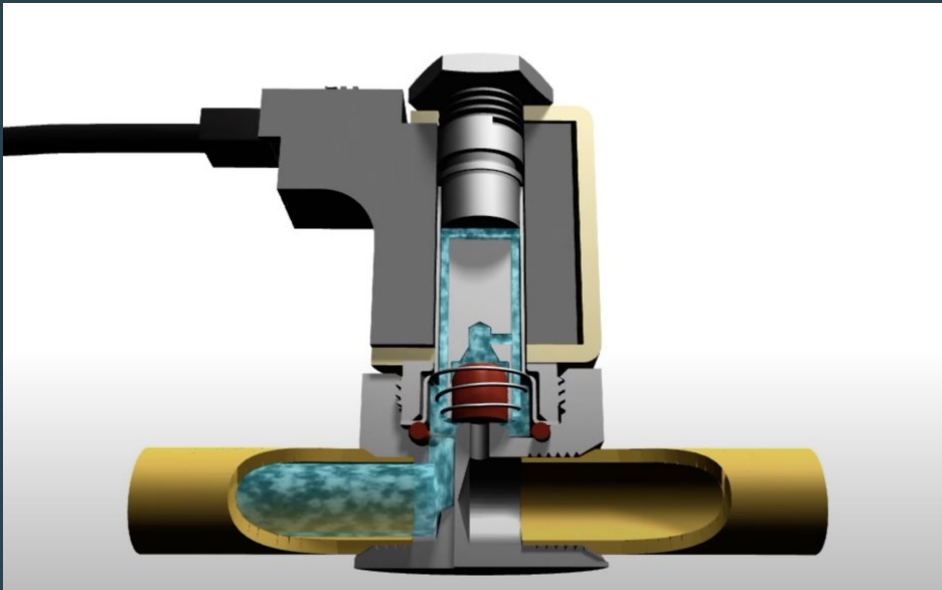
# Schematic of Valve Layout



V01-Normally Closed (Blue)  
V02-Normally Closed  
V03-Normally Open (Gray)  
V04-Normally Closed

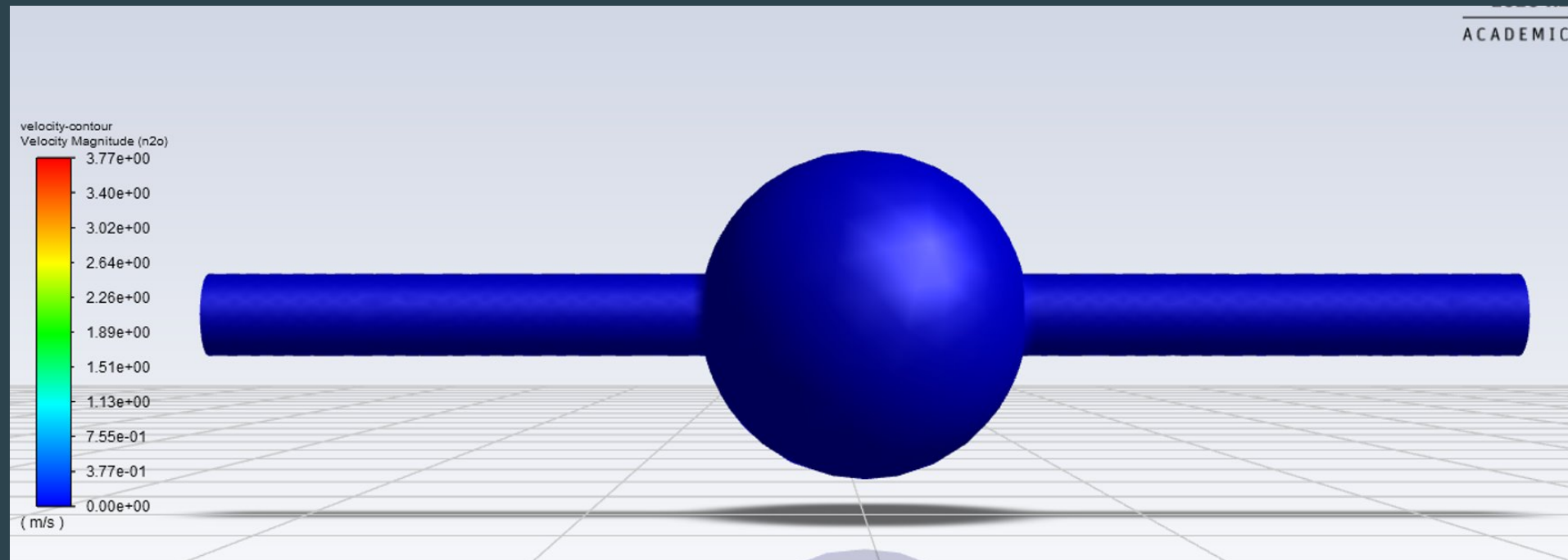
# Valve System Analysis

- CFD simulations performed on valves
- ANSYS and SimScale software was implemented
- Valve normally closed tested
- Inner and outer diameter 0.25 in.



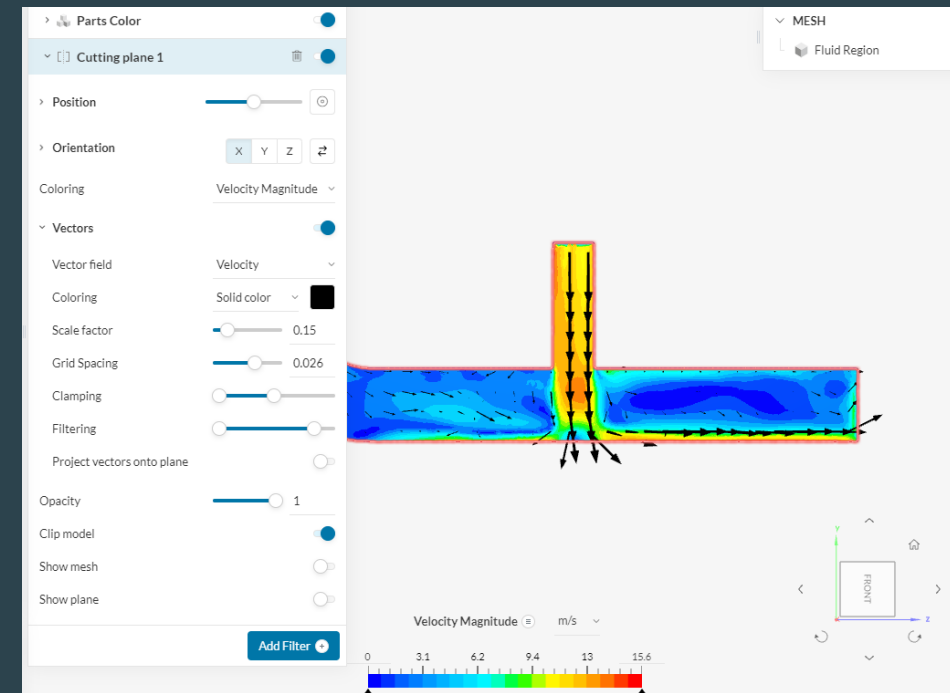
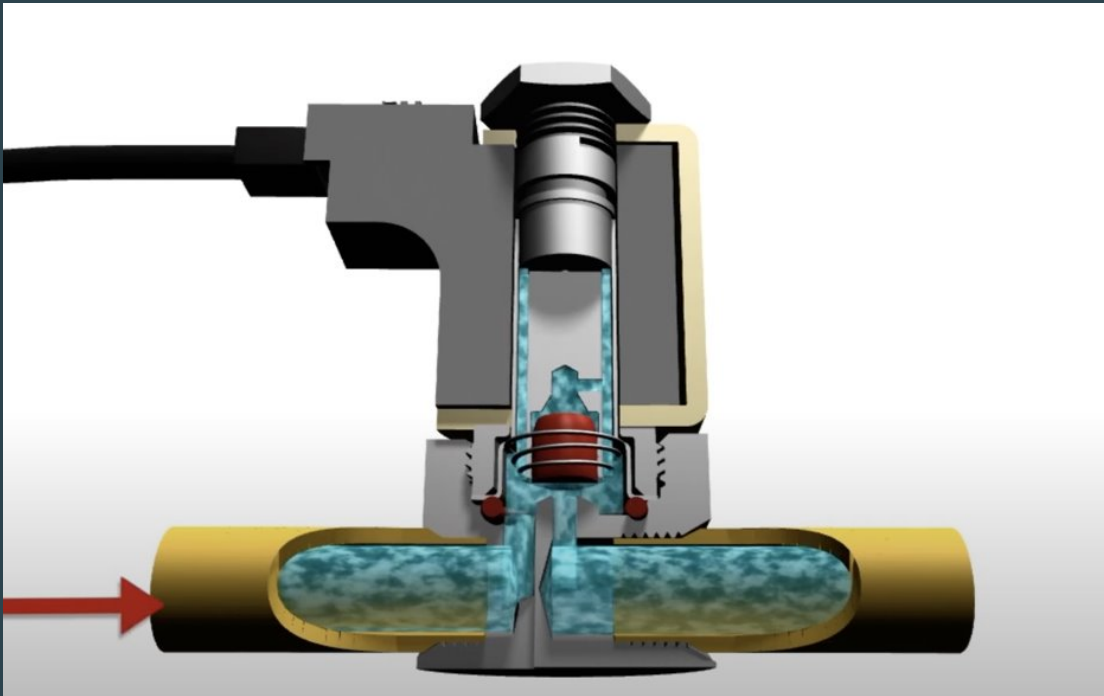
# Valve Analysis ANSYS

- Velocity entering valve 3.77 m/s
- Mass flow rate entering valve 3 kg/s



# Valve Analysis SimScale

- SimScale is a program able to understand CAD models
- Velocity at the stopper region is 9 m/s





# Base Structure Analysis



Vicente Estrada

- Structural Designer & Analyst

# Base Structure

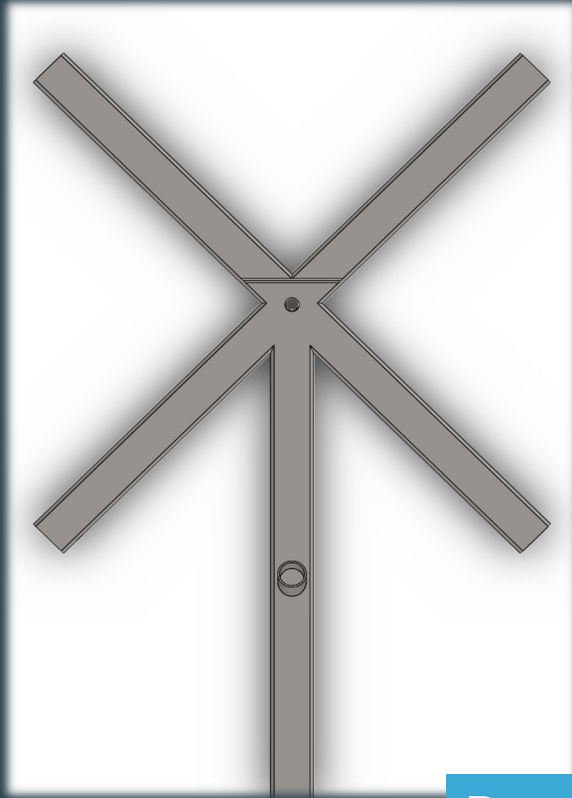
Revision 2



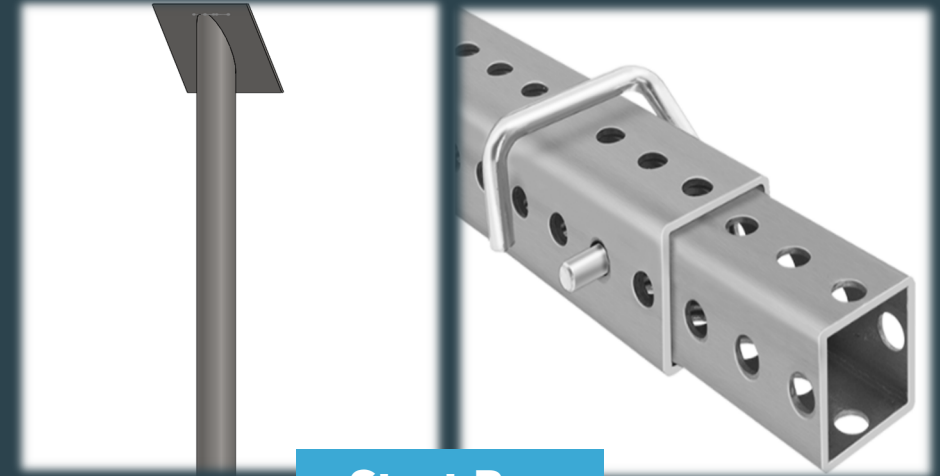
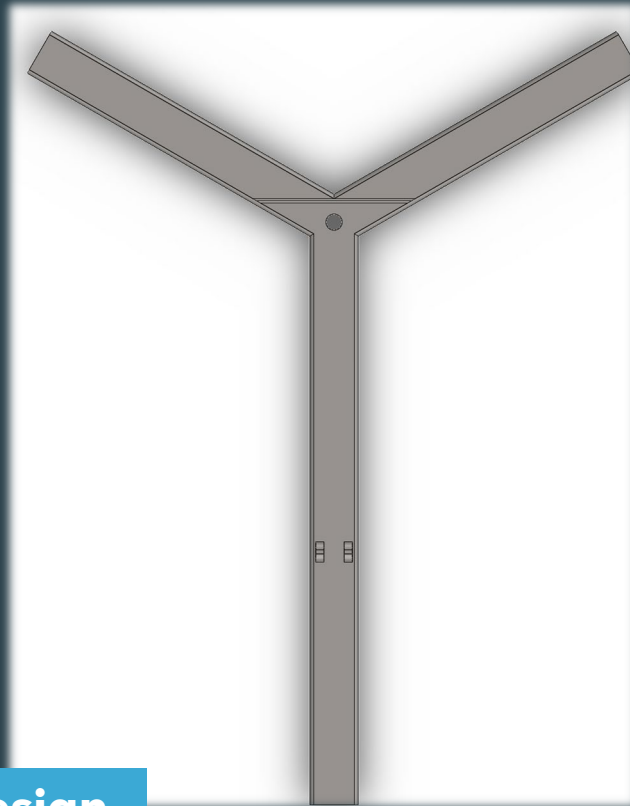
Final Revision



# Major Changes

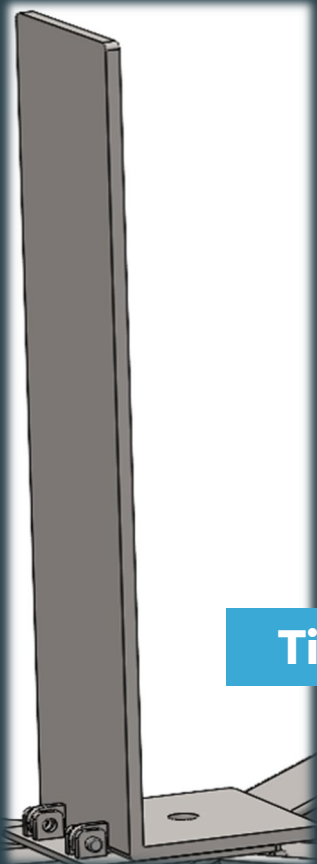


**Base Design**

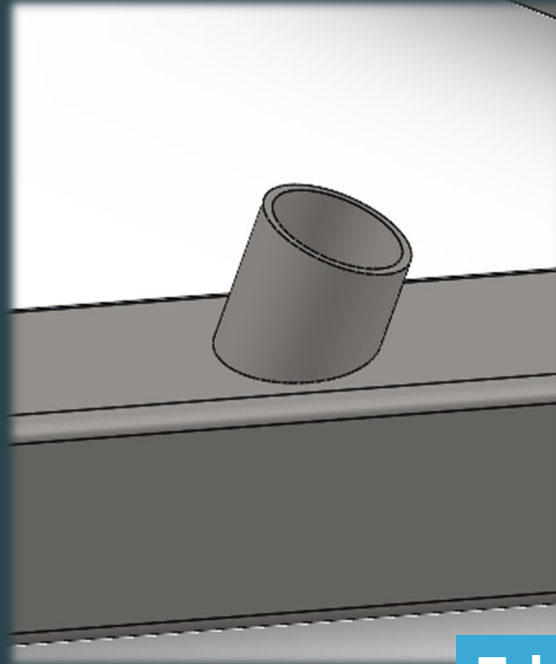
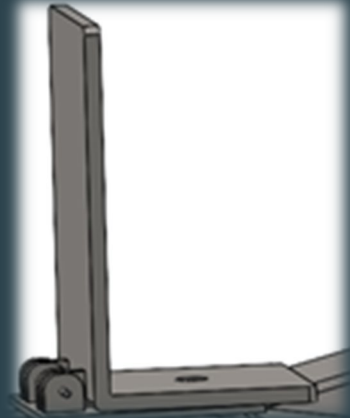


**Strut Bar**

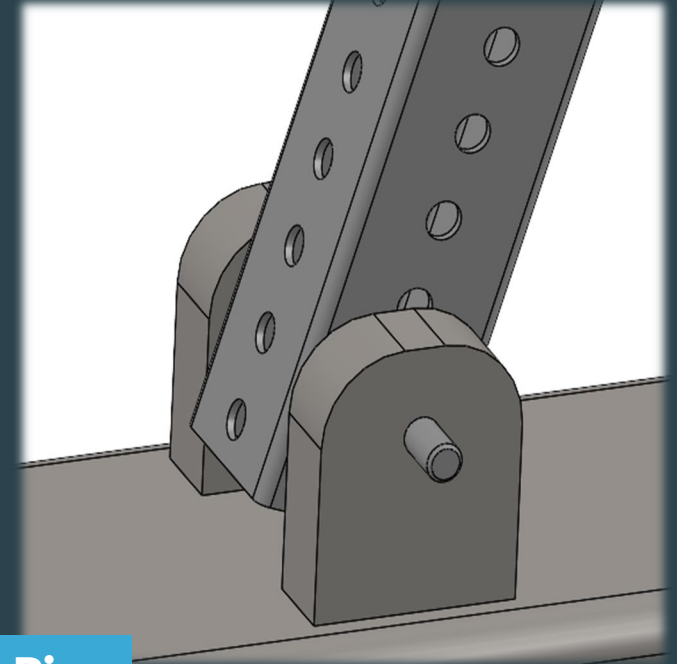
# Major Changes



Tilt Plate



Tabs & Pins



# Hand Calculations

$$W_T = 284.78N$$

$$F_T = 2500N$$

$$\Sigma F_y: 0 = R_s \cos(68) + R_y + F_T - W_T$$

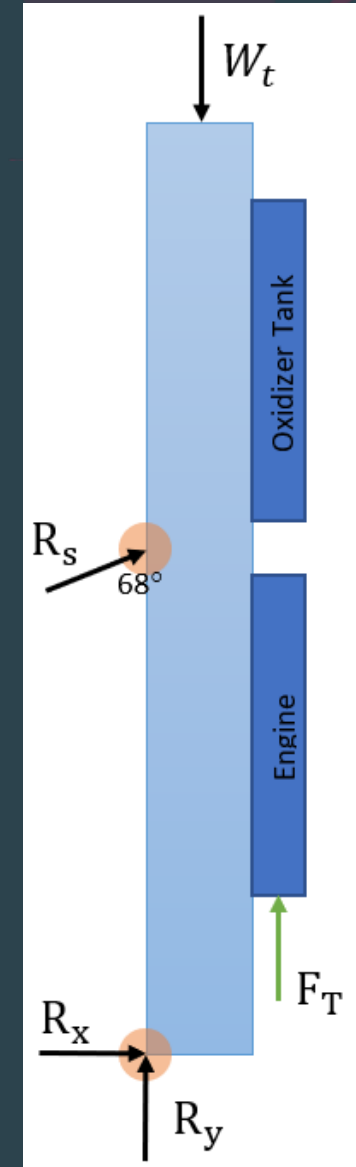
$$\Sigma F_x: 0 = R_s \sin(68) + R_x$$

$$\Sigma M: 0 = (0.3048m)F_T - (2.231m)R_s \sin(68) - 21.7Nm$$

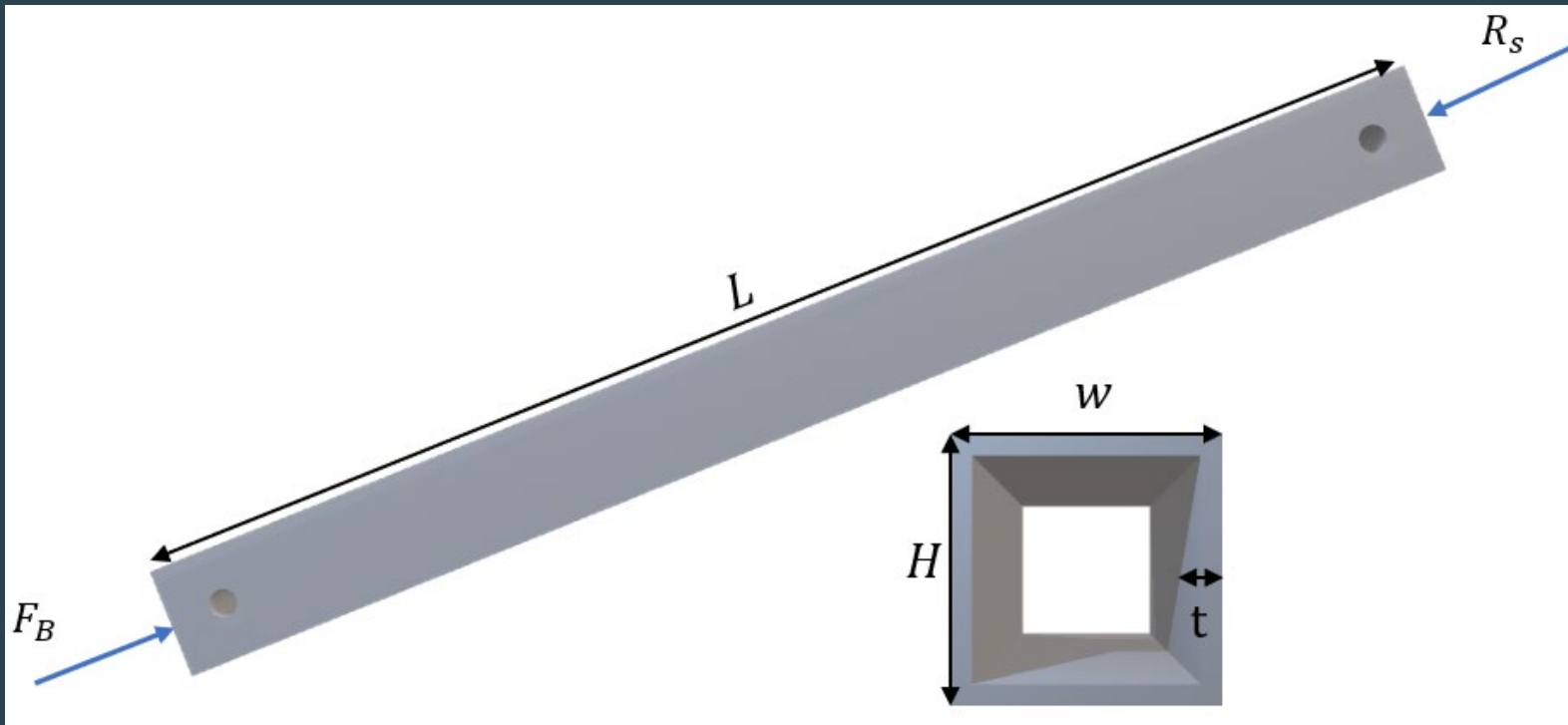
$$R_y = -2349.28N$$

$$R_x = -331.82N$$

$$R_s = 357.88N$$



# Hand Calculations



$$L = 2.4384 \text{ m}$$

$$H = w = 0.0635 \text{ m}$$

$$t = 2.67 \times 10^{-3} \text{ m}$$

$$k = 1$$

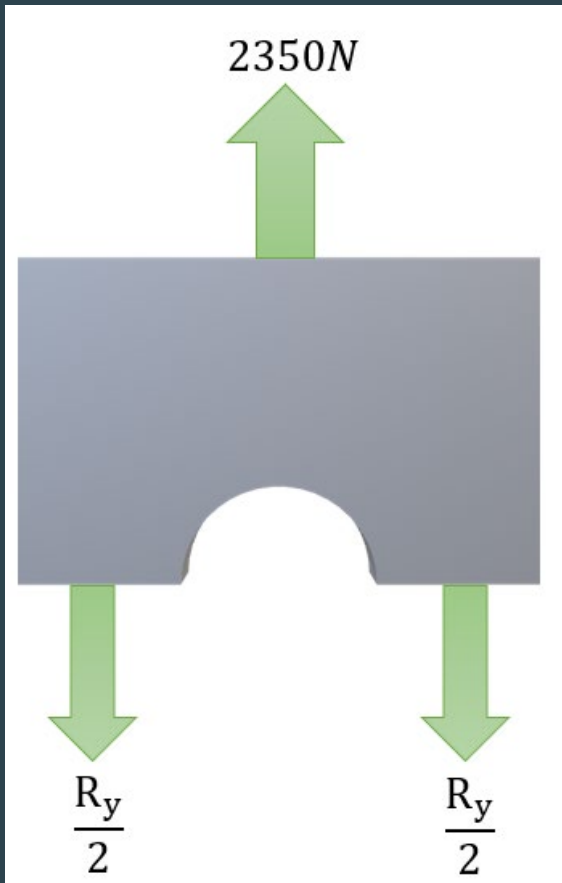
$$E = 200 \text{ GPa}$$

$$I = 39.9 \times 10^{-7} \text{ m}^4$$

$$P_{cr} = 127.83 \text{ kN}$$

$$R < P_{cr}$$

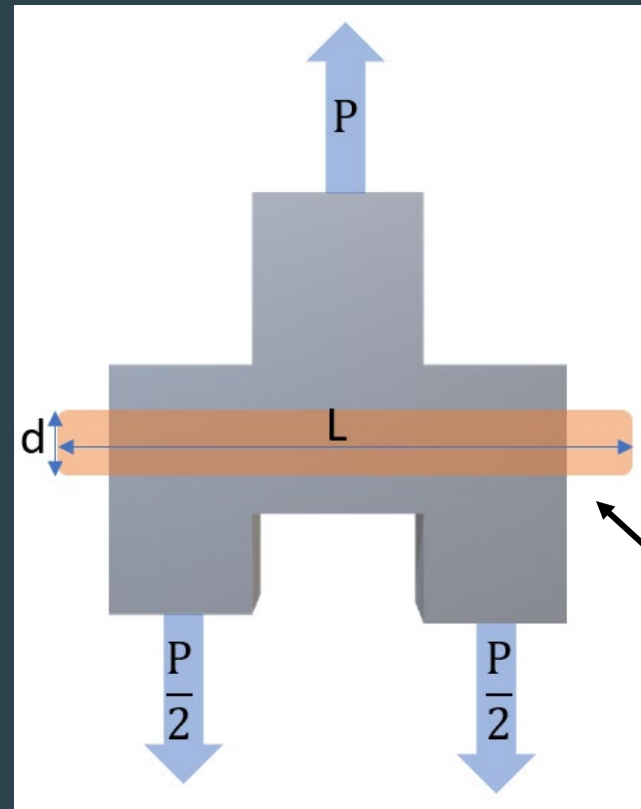
# Hand Calculations



$$\sigma_Y = 206 \text{ MPa}$$

$$\sigma = 2.33 \text{ MPa}$$

$$\sigma_Y > \sigma$$



$$P = R_y$$

$$\tau_U = 66278.5N$$

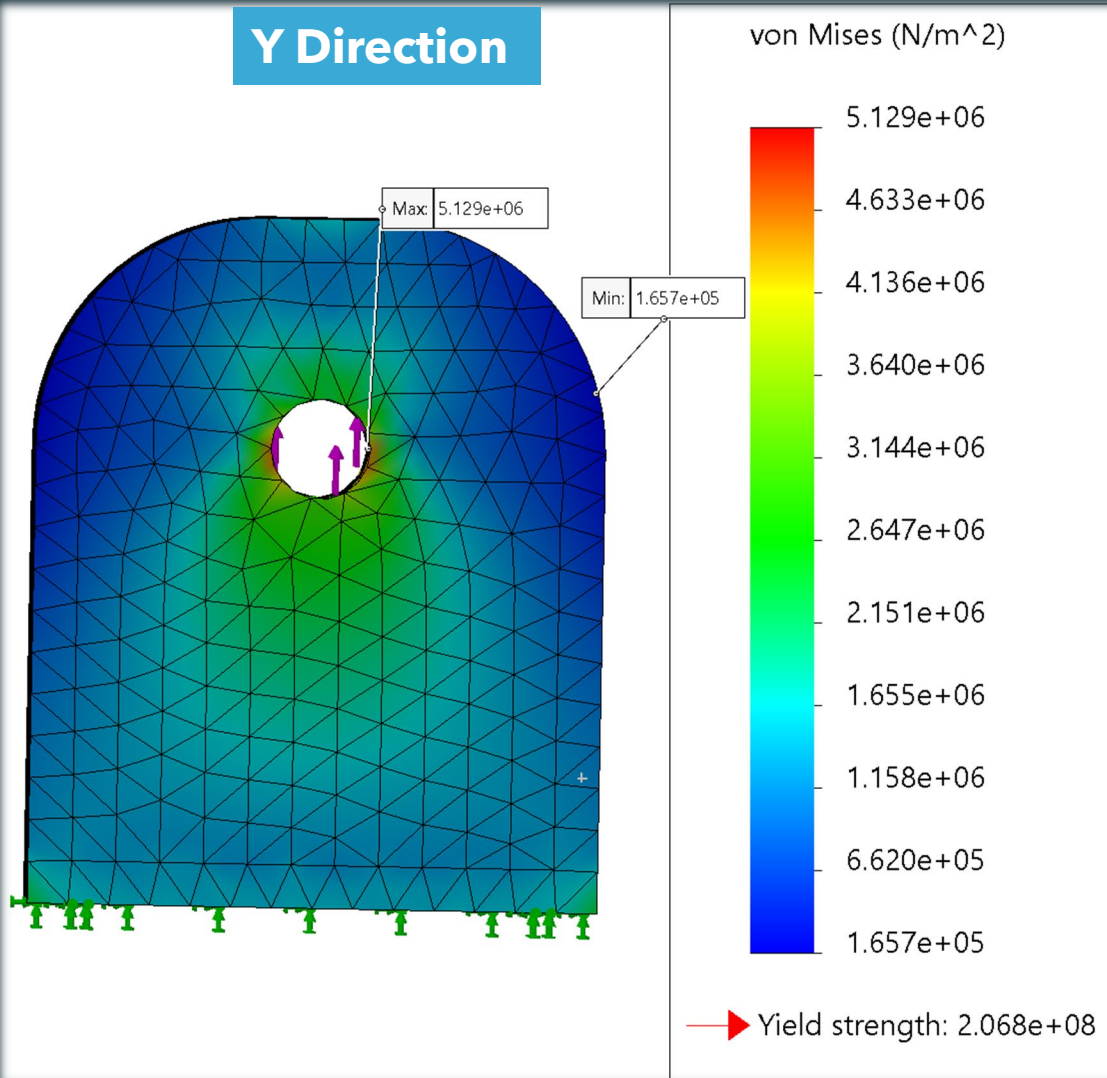
$$L = 0.127m$$

$$d = 0.4375m$$

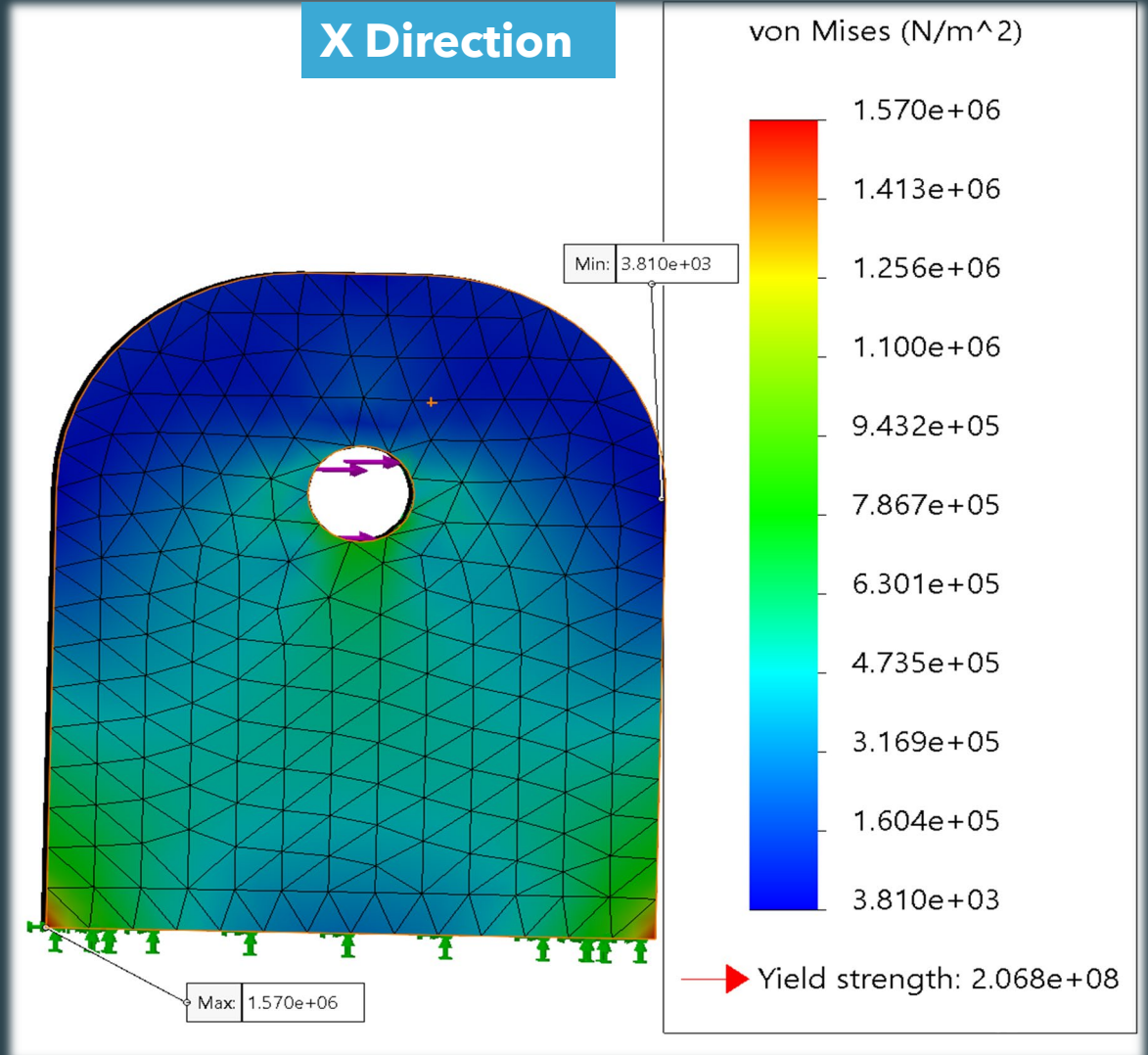
Stainless Steel Pin

# FEA

## Y Direction



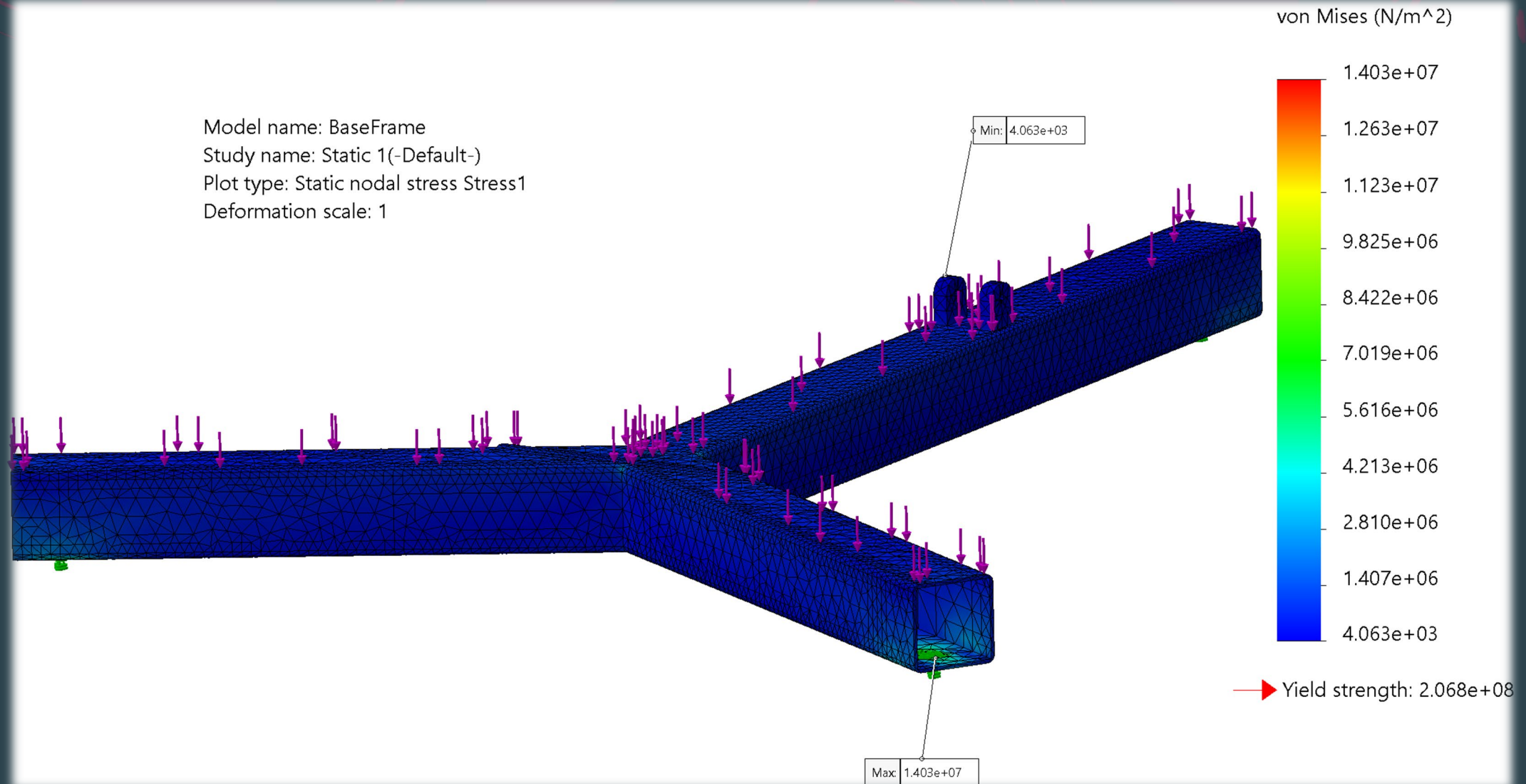
## X Direction



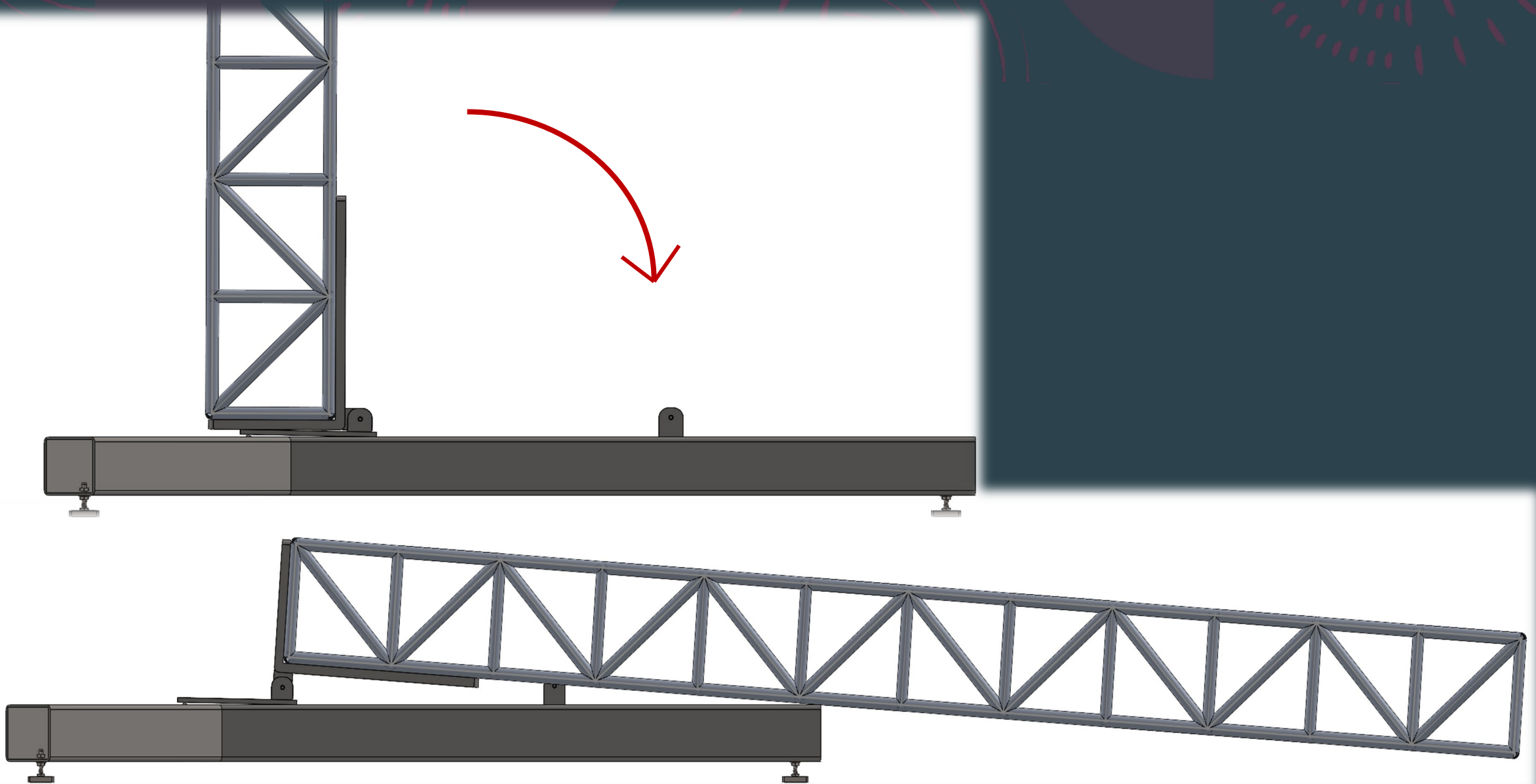


# FEA

Model name: BaseFrame  
Study name: Static 1(-Default-)  
Plot type: Static nodal stress Stress1  
Deformation scale: 1



# Transportation/Mounting Assembly



# Test Stand Analysis

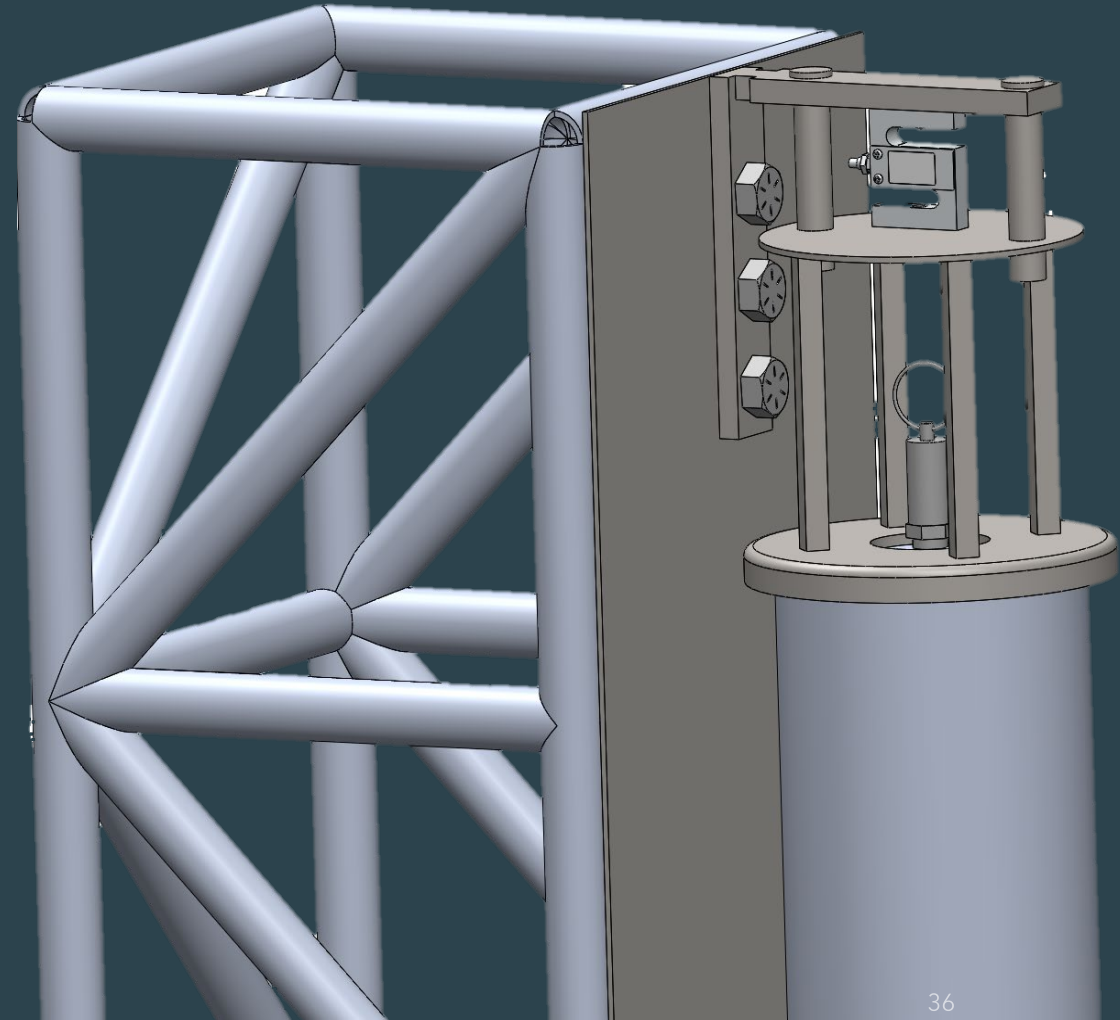


Alex Guilbaud

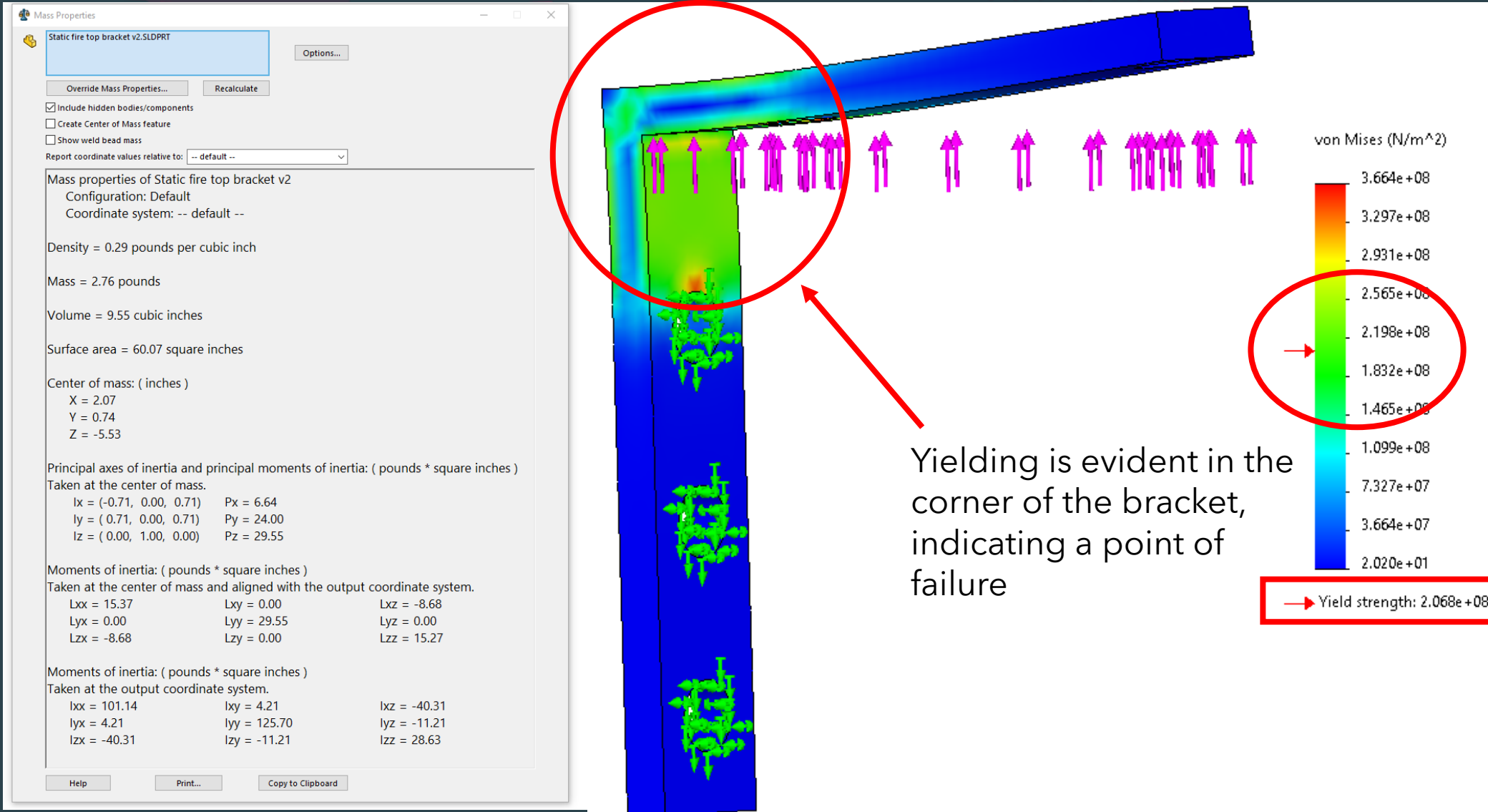
- Structural Designer & Analyst

# Mechanical Force Analysis

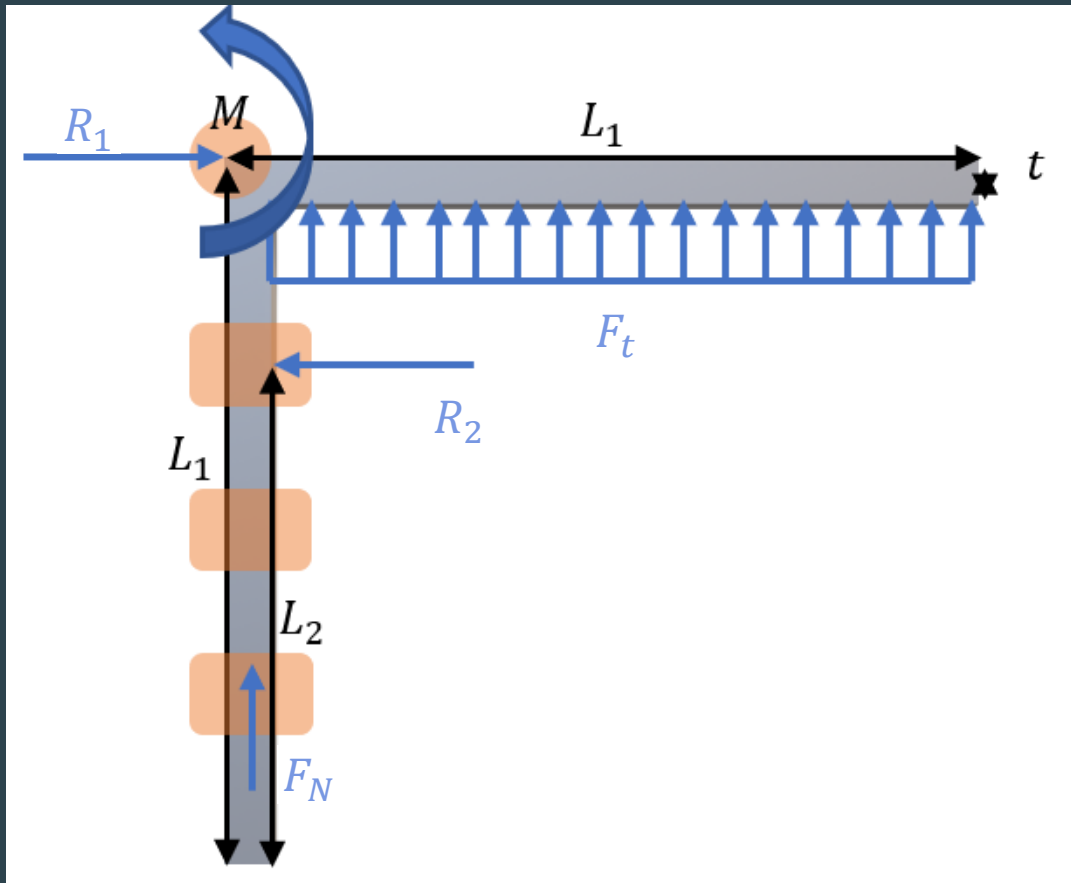
- Concern:
  - Original bracket has potential to fail
- Resolve:
  - Calculate bending stresses and conduct FEA; determine if true



# Mechanical Force Analysis



# Hand Calculations



$$L_1 = 0.191m$$

$$L_2 = 0.127m$$

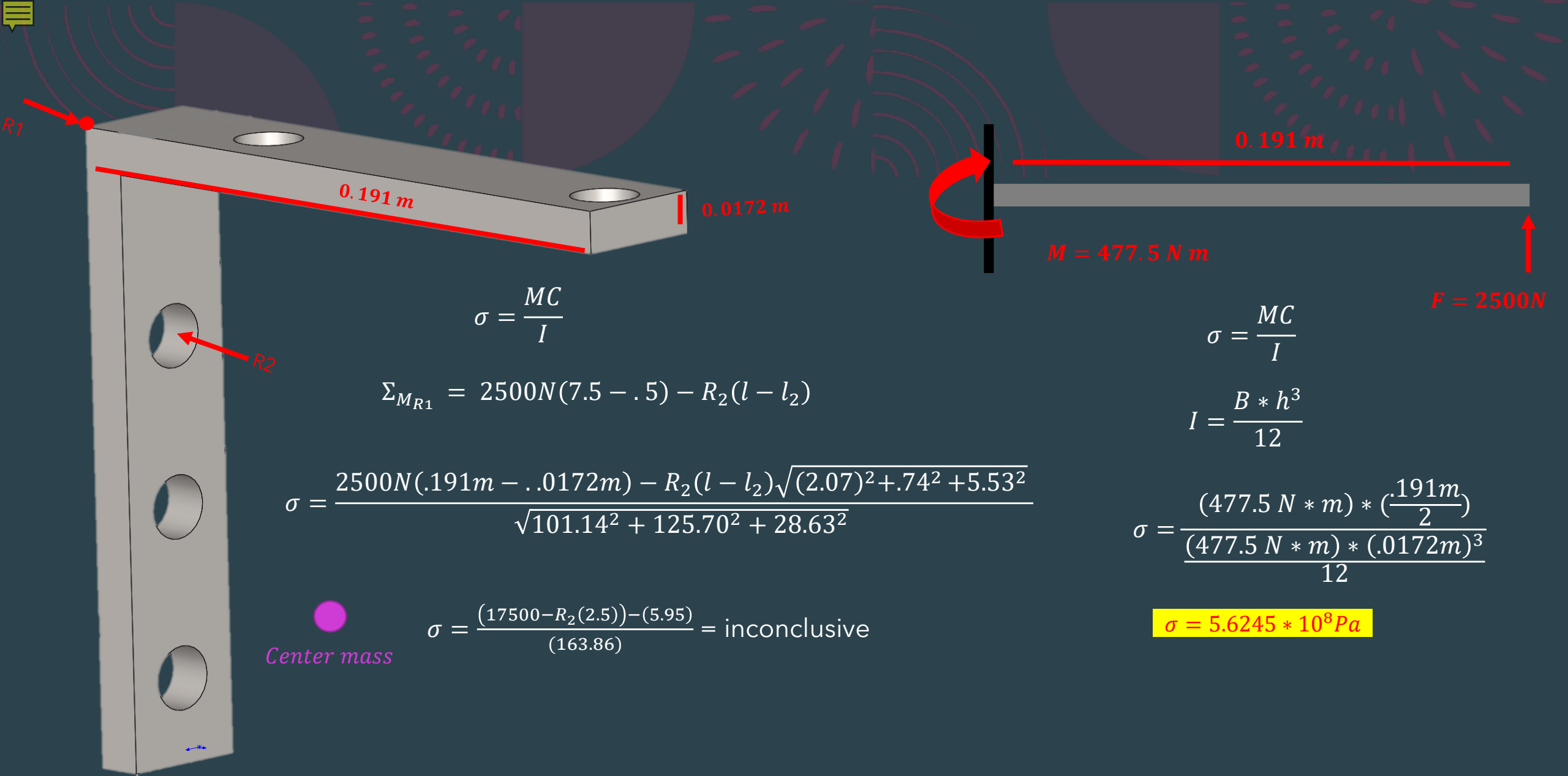
$$t = 0.0127m$$

$$F_T = 2500N$$

$$\Sigma F_x: R_1 = R_2$$

$$\Sigma F_y: F_n = -mg + F_T$$

$$\Sigma M_{R_1}: M = 17500Nm - R_2(L_1 - L_2)$$



$$\sigma = \frac{MC}{I}$$

$$\Sigma_{M_{R1}} = 2500N(7.5 - .5) - R_2(l - l_2)$$

$$\sigma = \frac{2500N(.191m - .0172m) - R_2(l - l_2)\sqrt{(2.07)^2 + .74^2 + 5.53^2}}{\sqrt{101.14^2 + 125.70^2 + 28.63^2}}$$

$$\sigma = \frac{(17500 - R_2(2.5)) - (5.95)}{(163.86)} = \text{inconclusive}$$

Center mass

$$\sigma = \frac{MC}{I}$$

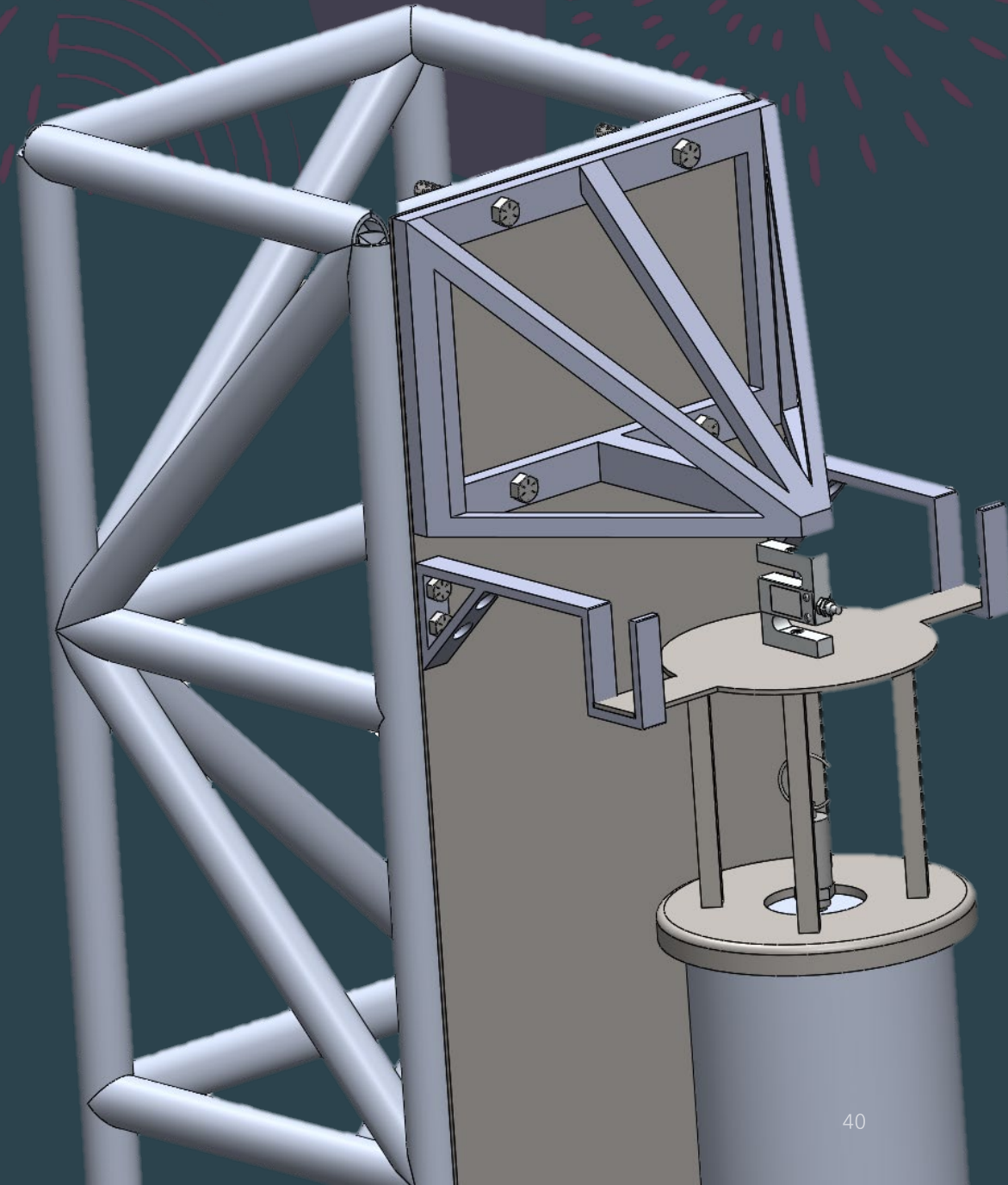
$$I = \frac{B * h^3}{12}$$

$$\sigma = \frac{(477.5 N * m) * (\frac{.191m}{2})}{\frac{(477.5 N * m) * (.0172m)^3}{12}}$$

$$\sigma = 5.6245 * 10^8 Pa$$

# Design Changes

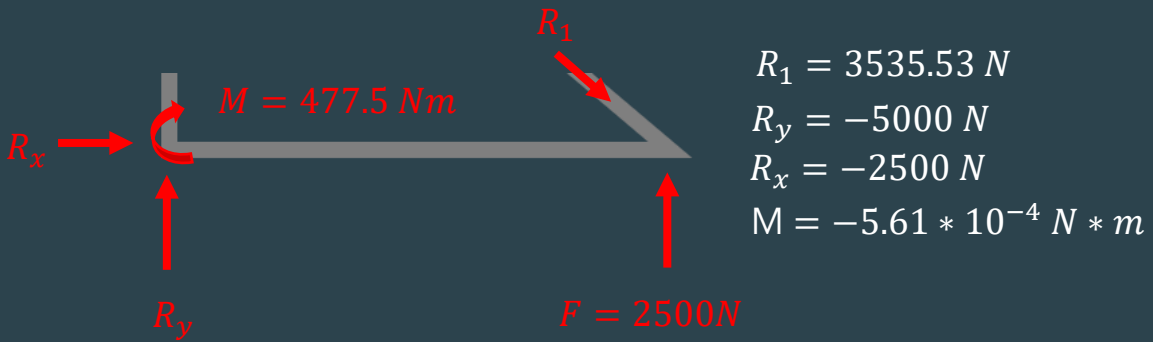
- Evidence of failure present; redesign necessary
- Reinforced truss shape adopted to prevent bending
- Water cut in 4 separate pieces and welded together





# Bracket FEA

- no yielding is evident with new design
- Same mounting hardware can be used

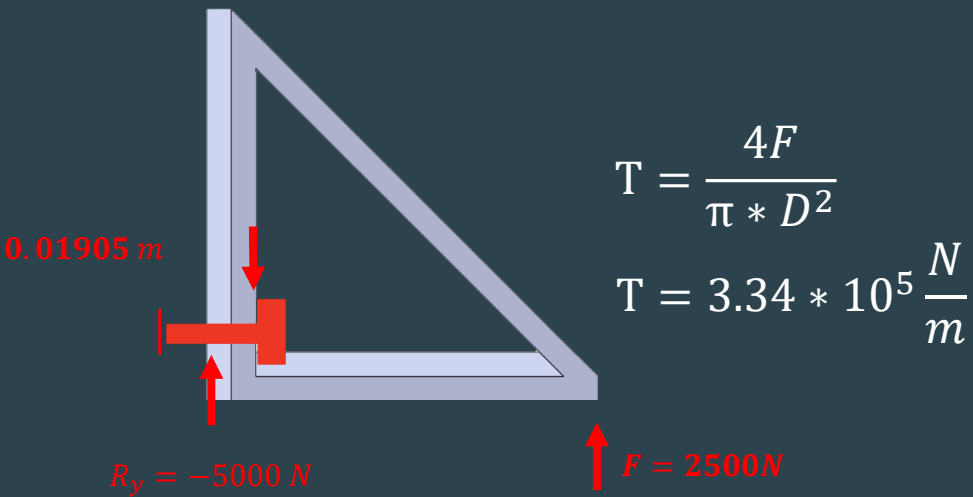


$$R_1 = 3535.53 \text{ N}$$

$$R_y = -5000 \text{ N}$$

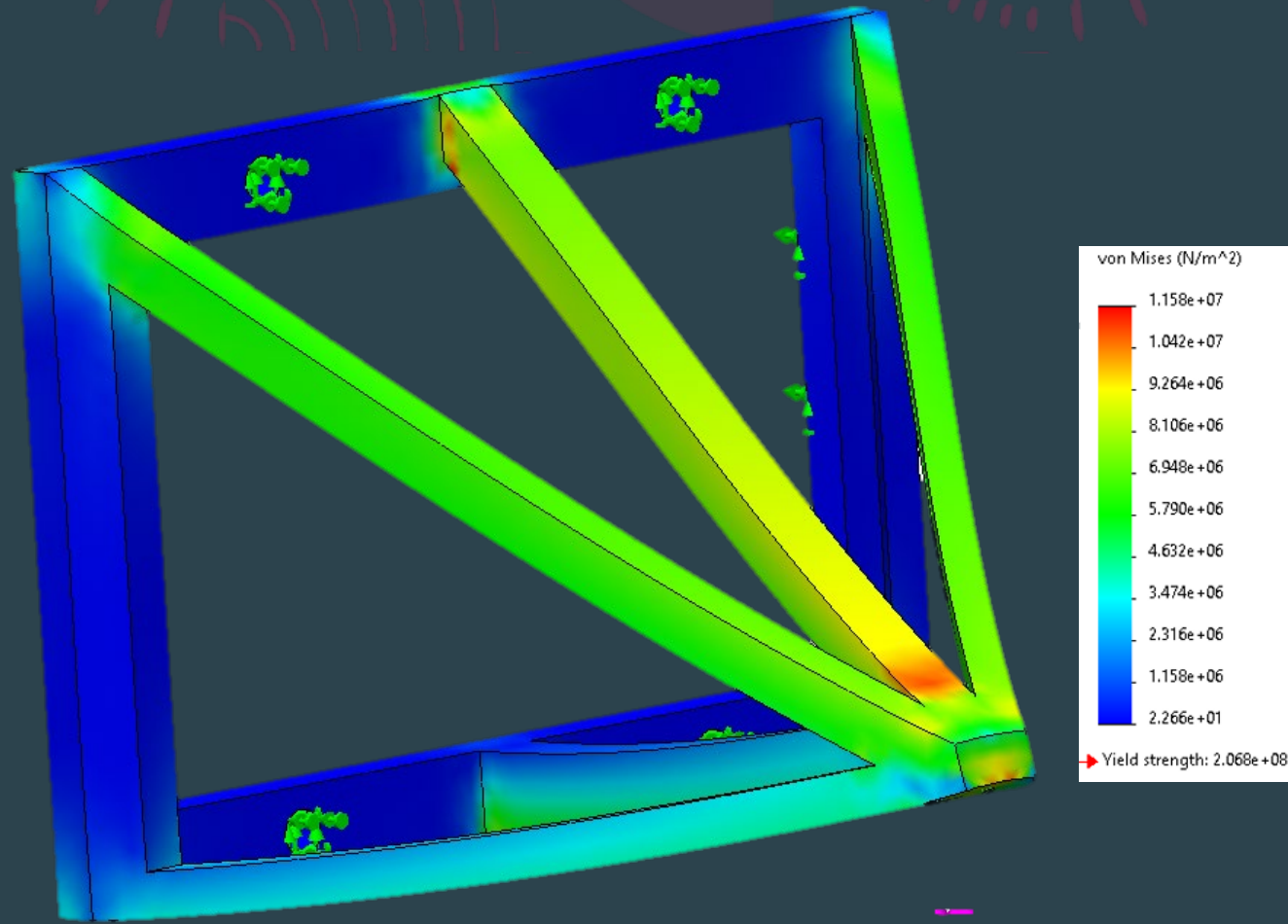
$$R_x = -2500 \text{ N}$$

$$M = -5.61 \times 10^{-4} \text{ N} \cdot \text{m}$$



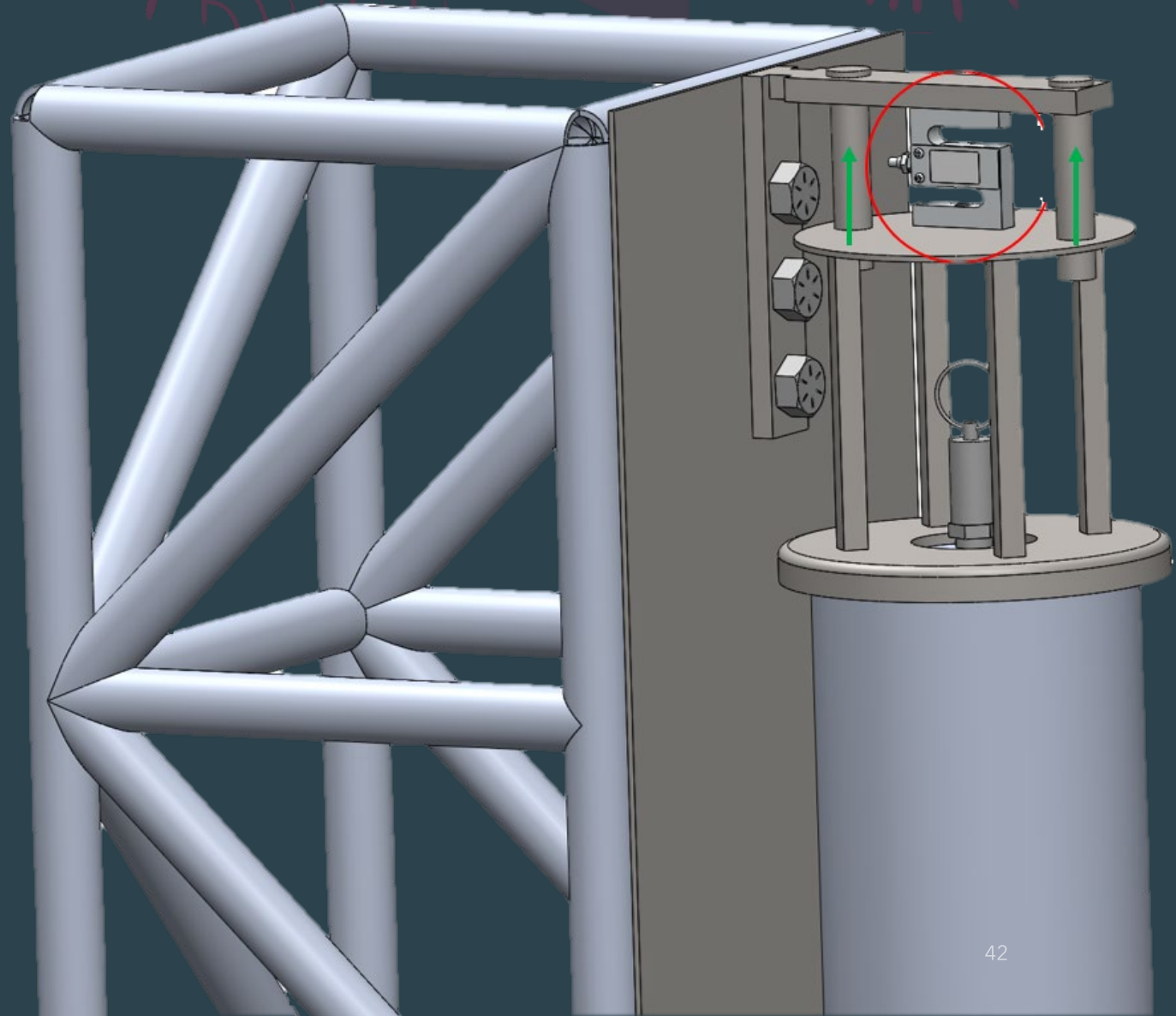
$$T = \frac{4F}{\pi * D^2}$$

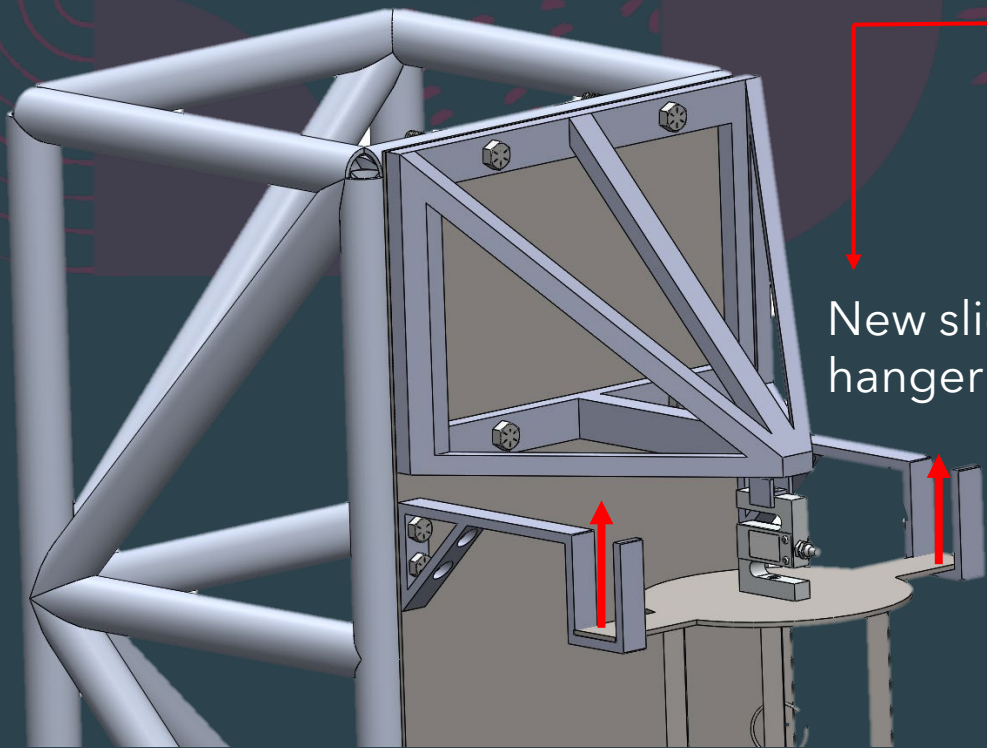
$$T = 3.34 * 10^5 \frac{\text{N}}{\text{m}}$$



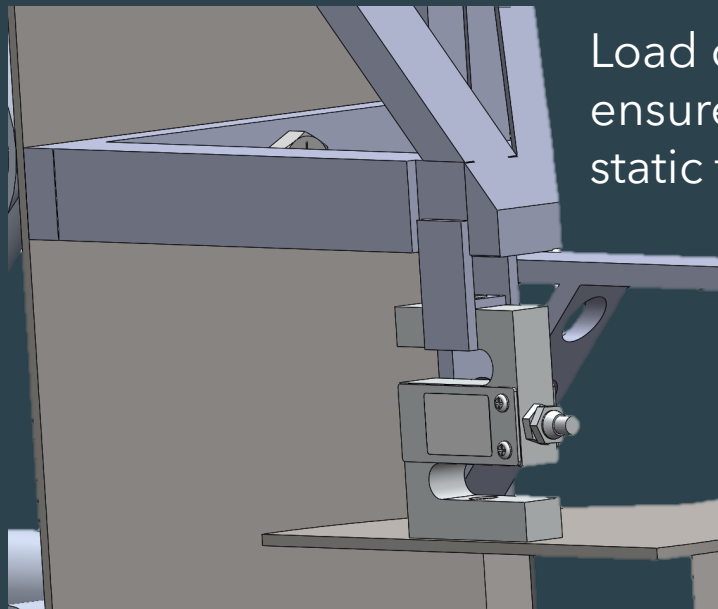
# Design changes

- Original slide system for load cell could bind/stuck during test fire
- Bracket redesign meant redesigning this slide system as well

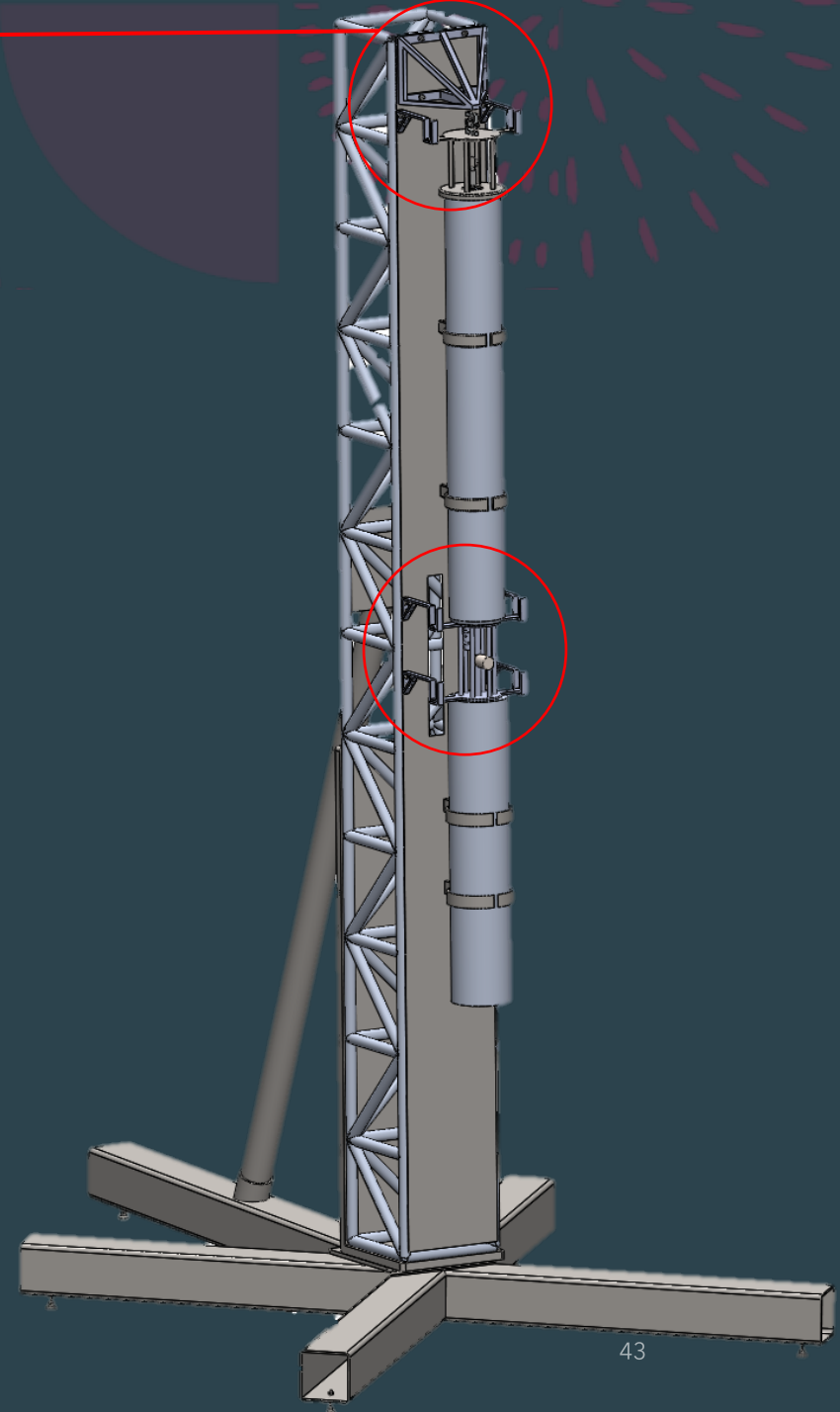




New slide system doubles as a hanger for motor assembly



Load cell is guided with tabs to ensure constant contact with static fire bracket



# Controls System



Anthony Urzua

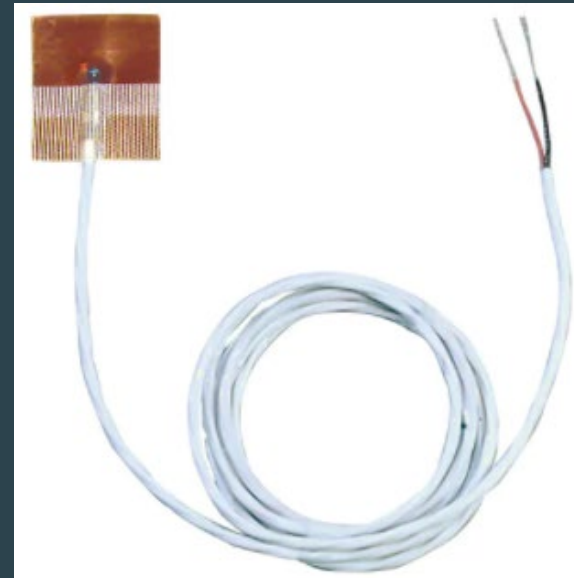
- Controls System Designer

# Control System Design

- Design a GUI to allow operators to control the fill, drain, and purging sequencing of the oxidizer tank using LabVIEW and Arduino

# Selected Sensors

- Explosion-Proof Industrial Pressure Transducer
- Self-Adhesive Polyimide Fast Response Thermistor Surface Sensor

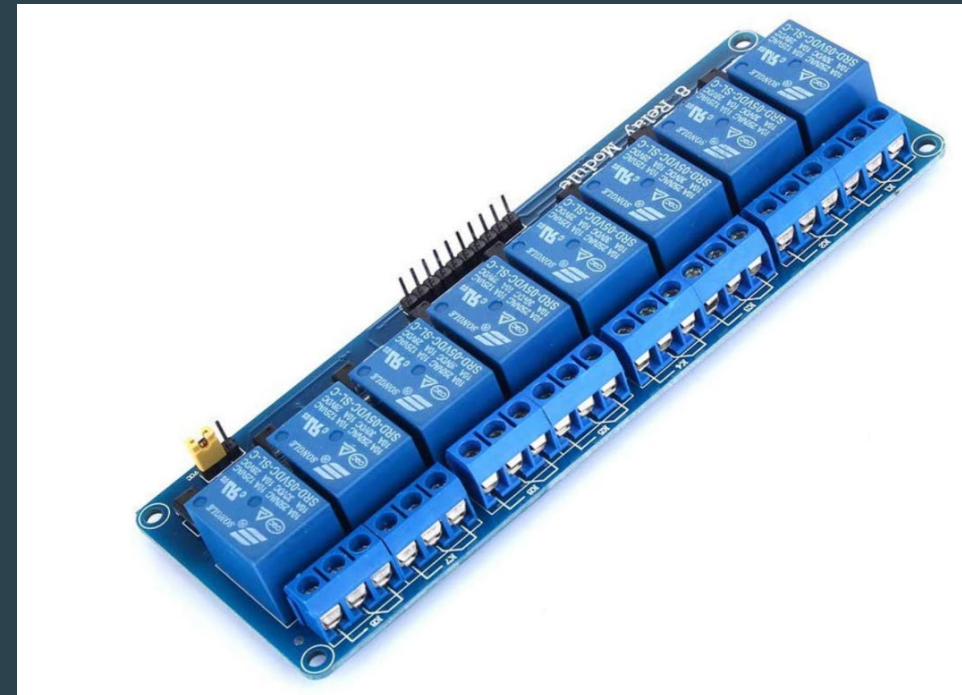


# Issues

- Pressure transducer was selected but was not able to be ordered. Could not get a hold of any technical documentation.

# Additional Hardware

- 6 V Battery
- 8-Channel 5V DC Relay Module Control Board

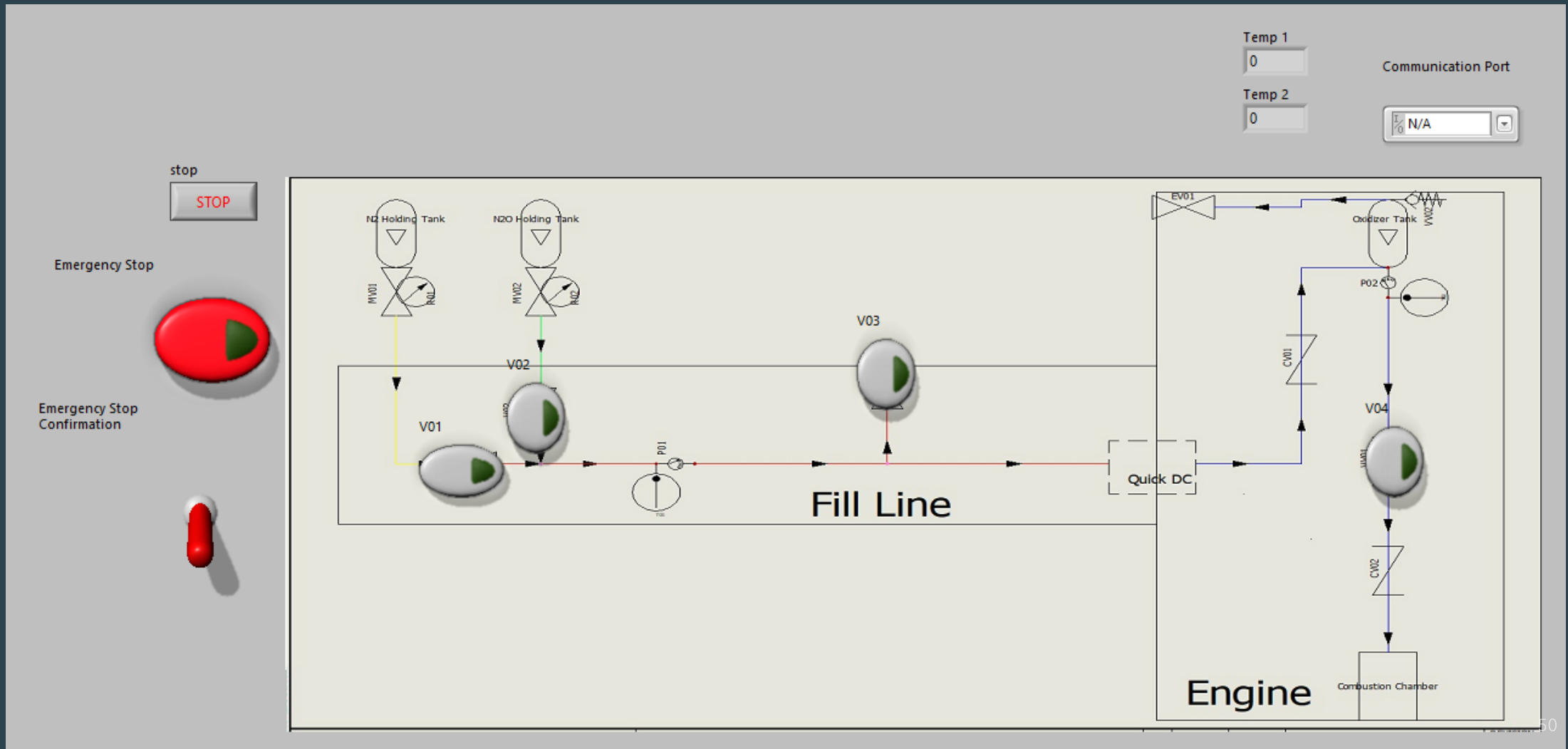




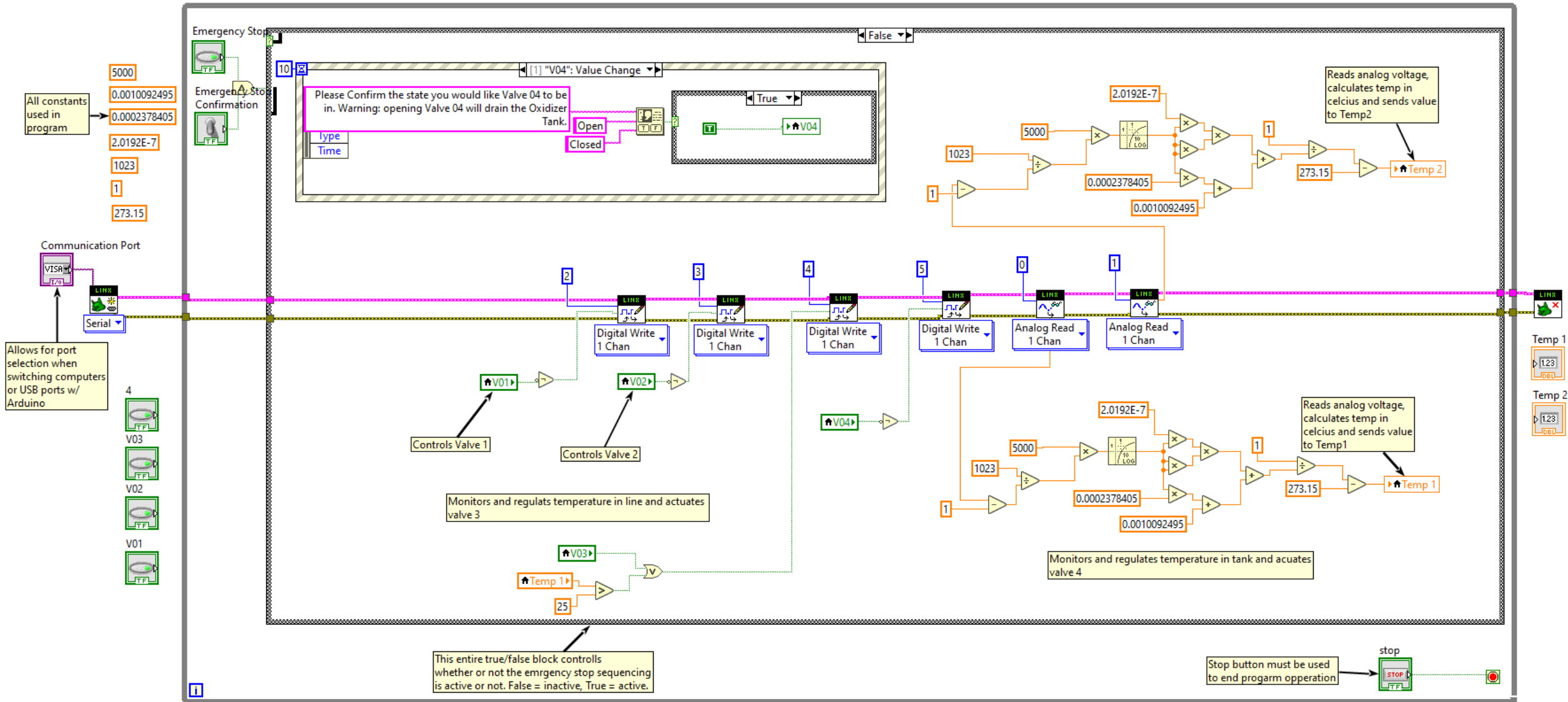
# Current Block Diagram



# Front Pannel

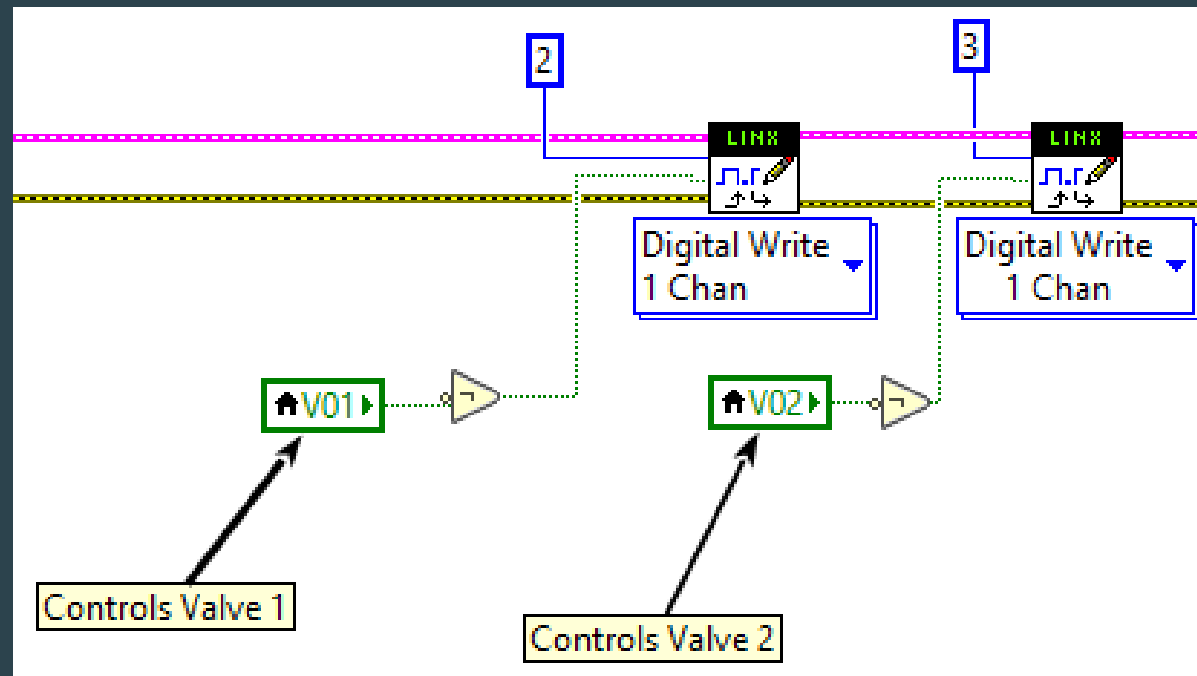


# Block Diagram



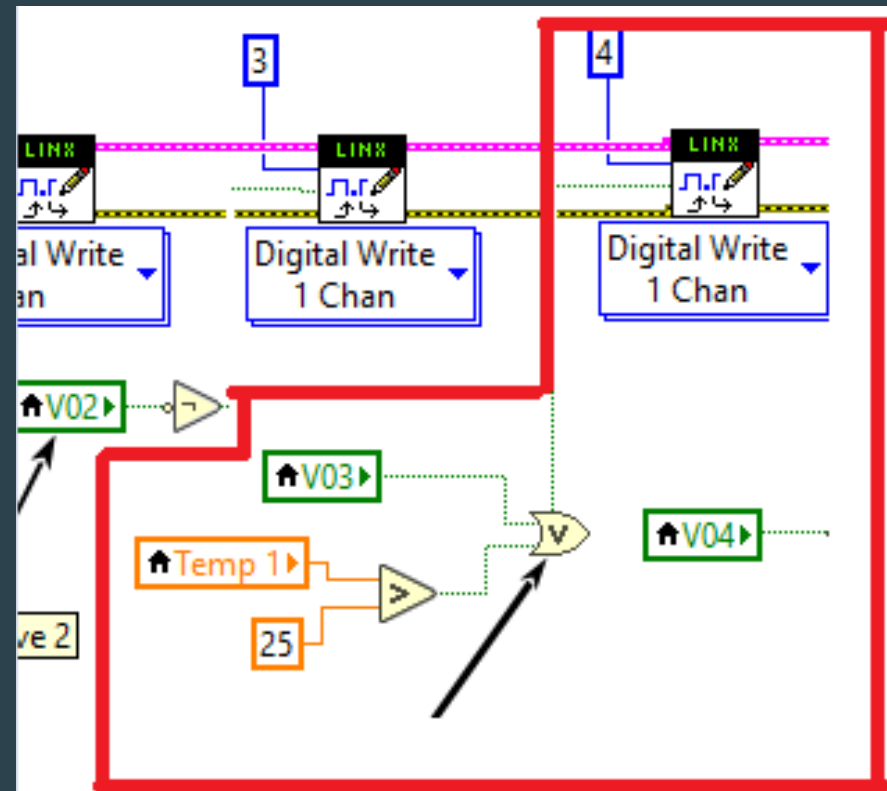
# Valve actuation logic

Normally Closed:

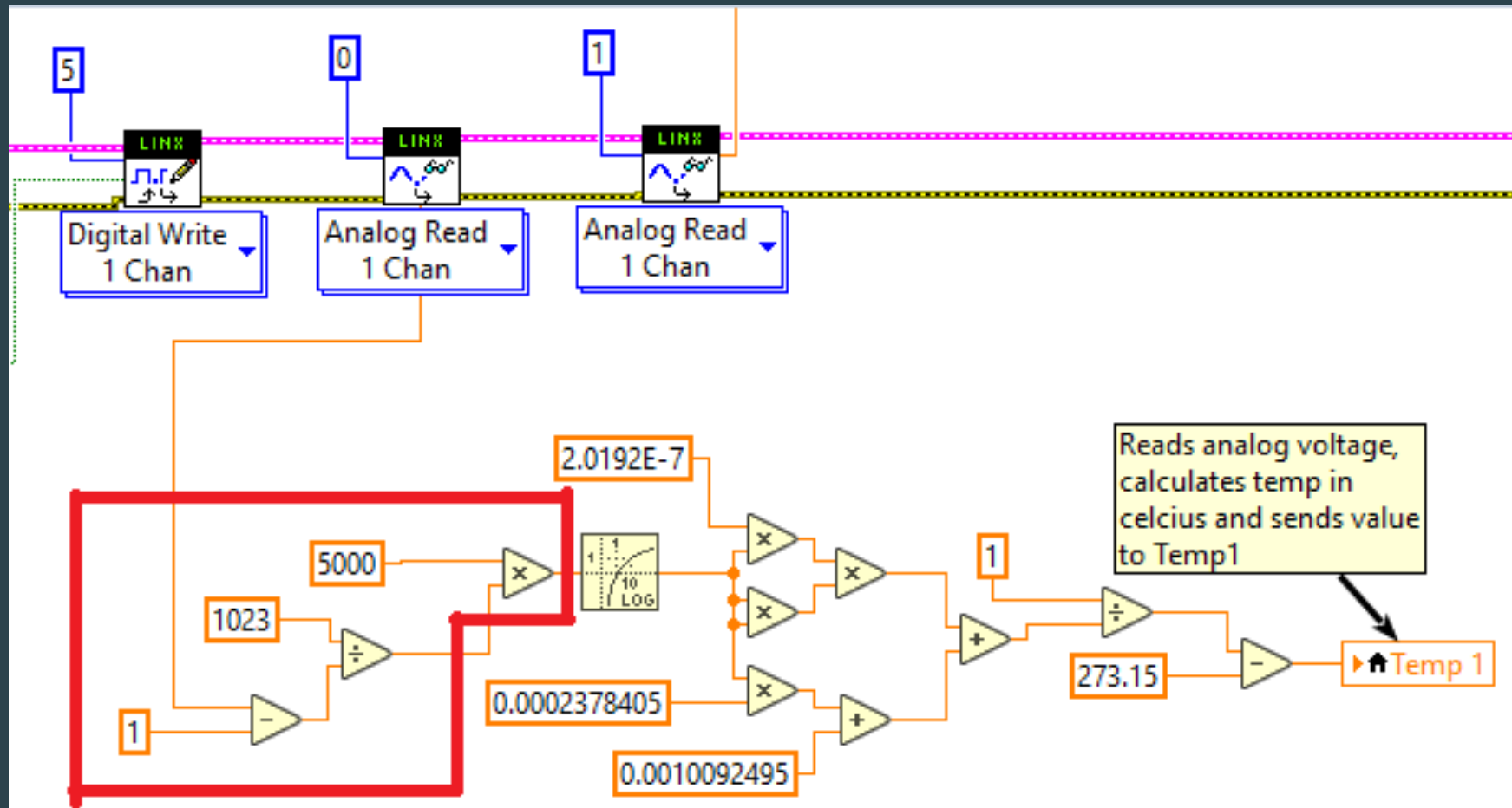


# Valve Actuation logic

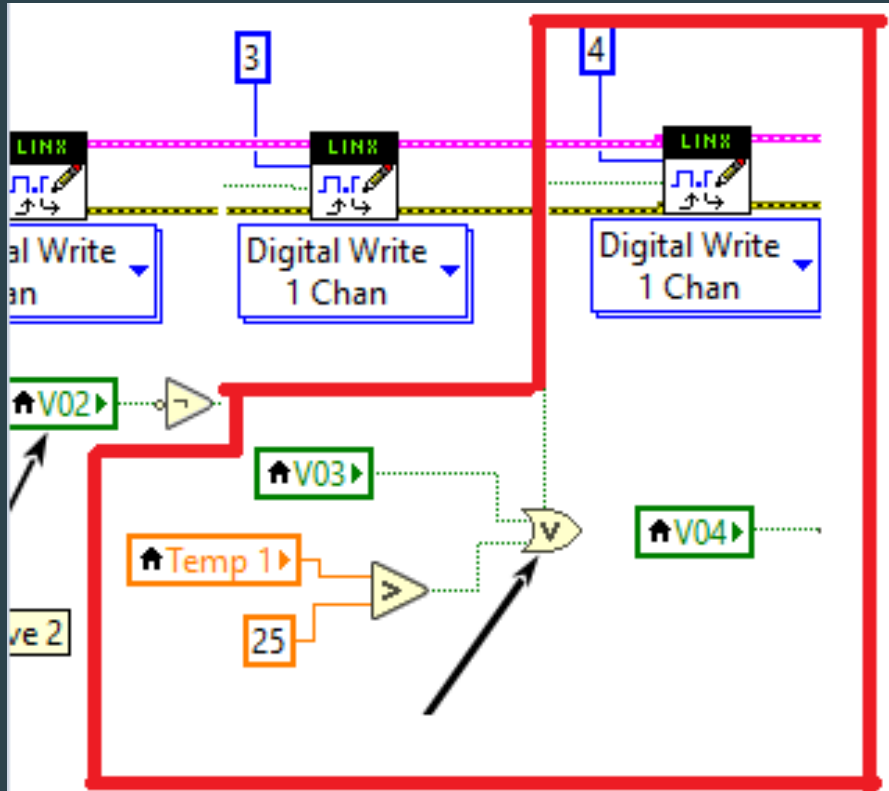
Normally Open:



# Temperature Calculation



# Temperature Regulation



Truth Table:

Temp	Valve controller	Temp    Valve	Valve Position
0	0	0	Closed
0	1	1	Open
1	0	1	Open
1	1	1	Open

# In Conclusion

- Python and ANSYS total times are within 2 minutes of each other.
- Valve losses increase total fill time.
- Structural designs reduce manufacturing complexity.
- Structure can withstand static fire forces.
- Controls system allows process to be done autonomously.



# Thank you!

- Dr. Santner
- ERSEDS
- GC Valves
- EMJ Metals
- College of ECST

# References

- [1] J. E. Zimmerman, B. S. Waxman, B. J. Cantwell and G. G. Zilliac, "Review and Evaluation of Models for Self-Pressurizing Propellant Tank Dynamics," in *49th AIAA/ASME/SAE/ASEE Joint Propulsion Conference*, San Jose, 2013.
- [2] N. G. Richter and B. H. Sage, "Thermal Conductivity of Fluids. Nitrous Oxide," *Journal of Chemical & Engineering Data*, vol. 8, no. 2, pp. 221-225, 1963.
- [3] M. Takahashi, N. Shibasaki-Kitakawa, C. Yokoyama and S. Takahashi, "Viscosity of Gaseous Nitrous Oxide from 298.15 K to 398.15 K at Pressures up to 25 MPa," *Journal of Chemical & Engineering Data*, vol. 41, no. 6, pp. 1495-1498, 1996.
- [4] ESDU, "Thermophysical properties of nitrous oxide," *ESDU Series on Physical Data, Chemical Engineering*, September 1991.
- [5] E. W. Lemmon and R. Span, "Short Fundamental Equations of State for 20 Industrial Fluids," *Journal of Chemical Engineering Data*, vol. 51, no. 3, pp. 785-850, 2006.